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分析具有奇異應力之功能梯度材料厚版(1/3)

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Abstract

Describing the behaviors of stress singularities correctly is essential for obtaining accurate numerical solutions of complicated problems with stress singularities. This analysis derives asymptotic solutions for functionally graded material (FGM) thick plates with geometrically induced stress singularities. The Reddy's plate theory is used to establish the equilibrium equations for FGM thick plates. Assuming the Young's modulus varies along the thickness and constant of Poisson's ratio, the eigenfunction expansion method is employed to the equilibrium equations in terms of displacement components for an asymptotic analysis in the vicinity of a sharp corner. The characteristic equations for determining the stress singularity order at the corner vertex and the corresponding corner functions are explicitly given for different combinations of boundary conditions along the radial edges forming the sharp corner. The non-homogeneous elasticity properties are present only in the characteristic equations corresponding to boundary conditions involving simple support. Finally, the effects of material non-homogeneity on the stress singularity orders are thoroughly examined by showing the minimum real values of the roots of the characteristic equations varying with the material properties and vertex angle.

Keywords: FGM thick plates, Stress singularities, Asymptotic solutions, Eigenfunction expansion method

摘要

為獲得複雜的應力奇異性問題的精確數值解，準確地描述應力奇異行為是非常重要的。本研究以 Reddy 板理論建構功能梯度材料(FGM)厚板之平衡方程式，並推導其幾何奇異應力之漸近解。本研究假設楊氏模數(Young's modulus)隨著厚度改變，而波松比(Poisson's ratio)則為常數。利用特徵函數展開法求解以位移分量表示之平衡方程式，求得尖銳角處之漸近解析解。推導對應於不同徑向邊界條件之特徵方程式以決定尖銳角處應力奇異階數，並進而獲得描述應力奇異行為之特徵函數。結果發現只有含簡支撐徑向邊界條件之特徵方程式包括非齊性的材料性質；但不管何種徑向邊界條件，材料的非齊性導致撓曲(bending)與拉伸(extension)行為互為耦合。

關鍵詞： 功能梯度厚板，應力奇異性，漸近解，特徵函數展開法

1. Introduction

Functionally graded materials (FGMs) were first produced in Japan in the mid-1980s (Niino and Maeda, 1990). An FGM is a multi-phase material comprised of different material components, such as ceramics and metals, that have various mixture ratios and microstructures. The gradual variation in material composition rather than sharp interfaces, as in the case of multilayered systems (i.e., laminated composites), significantly enhances the thermal and mechanical features of FGMs. Furthermore, FGMs can be designed to meet particular requirements, such as enhanced stiffness, toughness and resistance to corrosion, wear and high temperature, by using materials or material systems with various properties. Consequently, in the last two decades, FGMs have been used in numerous demanding engineering applications including military armor, thermal barrier coating for turbine blades and internal combustion engines and machine tools.

An FGM plate can be designed with material properties that vary gradually in the thickness direction, such that the plate is non-homogeneous in that direction only. A plate is a widely used component in practical engineering, and has various shapes. A plate with a reentrant corner or V-notch often shows stress singularities at the corner or vertex. These stress singularities must be considered if analysis is to be of real use. Stress singularity behaviors in a problem must be properly considered to obtain a convergent and accurate numerical solution (Leissa *et al.*, 1993; Huang *et al.*, 2005).

Studies of stress singularities resulting from various boundary conditions in angular corners of plates are limited to homogeneous plates or bi-material plates. Williams (1952a and 1952b) first showed the stress singularities of a thin plate under extension or bending due to various boundary conditions. Based on plane elasticity or classical plate theory, several studies used various approaches to investigate stress singularities at the interface corner of a bi-material plate (Hein and Erdogan, 1971; Bogy, 1971; Rao, 1971; Gdoutos and Theocaris, 1975; Dempsey and Sinclair, 1981; Ting and Chou, 1981). Burton and Sinclair (1986), Huang (2002a, 2002b, 2003, 2004), and McGee *et al.* (2005) investigated the stress singularities at thick plate corners using different plate theories or various analytical solution techniques. Based on three-dimensional elasticity, Bažant and Estenssoro (1977), Kerr and Parihar (1977), Schmitz *et al.* (1993), and Glushkov *et al.* (1999) applied different numerical solution techniques to investigate geometrically induced stress singularities at a three-dimensional vertex of a homogenous body. Somaratna and Ting (1986) and Ghahremani (1991) used a finite element approach to study stress singularities in anisotropic materials and composites. Huang and Leissa (2006) developed three-dimensional corner displacement functions for bodies of revolution.

Although geometrically induced stress singularities on an FGM plate have never been investigated, crack-related problems in FGMs have been frequently studied using plane or three-dimensional elasticity theory. Based on plane elasticity, Delale and Erdogan (1983) and Erdogan (1985) employed integral equation techniques to solve crack problems with mechanical loading, whereas Noda and Jin (1993) and Jin and Noda (1993) considered thermal loading. Furthermore, Erdogan and his coworkers (1988, 1991) investigated interface crack problems in bonded FGM plates. Using three-dimensional elasticity, Gu and Asaro (1997) applied an asymptotic solution of crack tip stress and displacement fields for homogeneous materials to examine small crack deflection in brittle FGMs. Gu *et al.* (1999), Rousseau and Tippur (2002), Kim and Paulino (2002), and Jin and Dodds Jr (2004) applied different finite element techniques to solve various crack problems. Chen *et al.* (2000) and Yue *et al.* (2003) utilized the mesh free Galerkin method and boundary element approach, respectively, to solve various crack problems.

This work examines geometrically induced stress singularities for an FGM plate using Reddy's thick plate theory. The equilibrium equations in terms of displacement components on the mid-plane are developed for an FGM thick plate. The in-plane displacement components are

coupled with the out-of-plane displacement component in the equations due to the non-homogeneity of an FGM. Asymptotic analysis of stress singularities in the vicinity of a sharp corner is carried out using the eigenfunction expansion approach. By assuming constant Poisson's ratio along the thickness, the characteristic equations for determining orders of stress singularity at the sharp corner are established for different sets of boundary conditions along the two radial edges forming the corner. The asymptotic solutions for the displacement functions are also explicitly presented. The effects of elasticity modulus variation along the thickness on stress singularity orders are thoroughly examined. These results are the first shown in the published literature.

2. Basic formulation

Consider a thick wedge (or sector plate) (Fig.1). The wedge is composed of FGM with material properties varying in the thickness direction (z direction). That is, the wedge is non-homogeneous only in the thickness direction. The displacement field for the wedge with cylindrical coordinates (Fig. 1), based on Reddy's third-order plate theory, is given as

$$\bar{u}(r, \theta, z) = u_0(r, \theta) + z[\psi_r(r, \theta) - \frac{4}{3}(\frac{z}{h})^2(\psi_r(r, \theta) + w_{,r}(r, \theta))], \quad (1a)$$

$$\bar{v}(r, \theta, z) = v_0(r, \theta) + z[\psi_\theta(r, \theta) - \frac{4}{3}(\frac{z}{h})^2(\psi_\theta(r, \theta) + \frac{1}{r}w_{,\theta}(r, \theta))], \quad (1b)$$

$$\bar{w} = w(r, \theta), \quad (1c)$$

where \bar{u} , \bar{v} , and \bar{w} are the displacement components in r , θ , and z directions, respectively; u_0 , v_0 , and w are the corresponding displacements on the mid-plane; ψ_r and ψ_θ are the rotations of the mid-plane normal in the radial and circumferential directions, respectively; the subscript “ j ” denotes partial differentials with respect to the independent variable j . This displacement field leads to zero shear stresses, σ_{zr} and $\sigma_{z\theta}$, on the plate top and bottom surfaces.

Introducing the following stress resultants

$$\begin{Bmatrix} Q_\beta \\ R_\beta \end{Bmatrix} = \int_{-h/2}^{h/2} \sigma_{\beta z} \begin{Bmatrix} 1 \\ z \end{Bmatrix} dz, \quad (2a)$$

$$\begin{Bmatrix} N_\beta \\ M_\beta \\ P_\beta \end{Bmatrix} = \int_{-h/2}^{h/2} \sigma_{\beta\beta} \begin{Bmatrix} 1 \\ z \\ z^3 \end{Bmatrix} dz, \quad (2b)$$

$$\begin{Bmatrix} N_{r\theta} \\ M_{r\theta} \\ P_{r\theta} \end{Bmatrix} = \int_{-h/2}^{h/2} \sigma_{r\theta} \begin{Bmatrix} 1 \\ z \\ z^3 \end{Bmatrix} dz \quad (2c)$$

where σ_{ij} are stress components, and using the principle of stationary potential energy, one can obtain the equilibrium equations without external loading (cf. Reddy, 1999)

$$N_{r,r} + N_{r\theta,\theta}/r + (N_r - N_\theta)/r = 0, \quad (3a)$$

$$N_{r\theta,r} + N_{\theta,\theta}/r + 2N_{r\theta}/r = 0, \quad (3b)$$

$$C_1(P_{r,rr} + \frac{2}{r}P_{r,r} + \frac{1}{r^2}P_{\theta,\theta\theta} - \frac{1}{r}P_{\theta,r} + \frac{2}{r}P_{r\theta,r\theta} + \frac{2}{r^2}P_{r\theta,\theta}) + \frac{\bar{Q}_r}{r} + \bar{Q}_{r,r} + \frac{1}{r}\bar{Q}_{\theta,\theta} = 0, \quad (3c)$$

$$\bar{M}_{r,r} + \frac{\bar{M}_r}{r} - \frac{\bar{M}_\theta}{r} + \frac{1}{r}\bar{M}_{r\theta,\theta} - \bar{Q}_r = 0, \quad (3d)$$

$$\frac{1}{r}\bar{M}_{\theta,\theta} + \bar{M}_{r\theta,r} + \frac{2\bar{M}_{r\theta}}{r} - \bar{Q}_\theta = 0, \quad (3e)$$

where $C_1 = \frac{4}{3h^2}$, $C_2 = \frac{4}{h^2}$, $\bar{M}_{r\theta} = M_{r\theta} - C_1 P_{r\theta}$, $\bar{M}_\beta = M_\beta - C_1 P_\beta$, $\bar{Q}_\beta = Q_\beta - C_2 R_\beta$, h is the thickness of plate and subscript β denotes r or θ . In addition, the following boundary conditions along $\theta = \alpha$ should be specified:

$$\begin{aligned} & u_0 \text{ or } N_{r\theta}, \quad v_0 \text{ or } N_\theta, \quad \psi_\theta \text{ or } \bar{M}_\theta, \quad \psi_r \text{ or } \bar{M}_{r\theta}, \\ & w \text{ or } \bar{Q}_\theta + C_1\left(\frac{2}{r}P_{r\theta} + 2P_{r\theta,r} + \frac{1}{r}P_{\theta,\theta}\right), \text{ and } \frac{w_{,\theta}}{r} \text{ or } P_\theta. \end{aligned} \quad (4a)$$

The circumferential boundary conditions (at $r=R$) should prescribe:

$$\begin{aligned} & u_0 \text{ or } N_r, \quad v_0 \text{ or } N_{r\theta}, \quad \psi_\theta \text{ or } \bar{M}_{r\theta}, \quad \psi_r \text{ or } \bar{M}_r, \\ & w \text{ or } \bar{Q}_r + C_1\left(\frac{P_r}{r} + P_{r,r} + \frac{2}{r}P_{r\theta,\theta} - \frac{P_\theta}{r}\right), \text{ and } w_{,r} \text{ or } P_r. \end{aligned} \quad (4b)$$

For an isotropic and elastic plate, the relationships between the stress resultants and displacement components are established by using strain-displacement and stress-strain relationships. They are,

$$\begin{aligned} N_r = & \frac{\bar{D}_0}{r}u_0 + \bar{E}_0u_{0,r} + \frac{\bar{D}_0}{r}v_{0,\theta} - \frac{C_1\bar{D}_3}{r}w_{,r} - C_1\bar{E}_3w_{,rr} - \frac{C_1\bar{D}_3}{r^2}w_{,\theta\theta} \\ & + \frac{1}{r}(\bar{D}_1 - C_1\bar{D}_3)\psi_r + (\bar{E}_1 - C_1\bar{E}_3)\psi_{r,r} + \frac{1}{r}(\bar{D}_1 - C_1\bar{D}_3)\psi_{\theta,\theta}, \end{aligned} \quad (5a)$$

$$\begin{aligned} N_\theta = & \frac{\bar{E}_0}{r}u_0 + \bar{D}_0u_{0,r} + \frac{\bar{E}_0}{r}v_{0,\theta} - \frac{C_1\bar{E}_3}{r}w_{,r} - C_1\bar{D}_3w_{,rr} - \frac{C_1\bar{E}_3}{r^2}w_{,\theta\theta} \\ & + \frac{1}{r}(\bar{E}_1 - C_1\bar{E}_3)\psi_r + (\bar{D}_1 - C_1\bar{D}_3)\psi_{r,r} + \frac{1}{r}(\bar{E}_1 - C_1\bar{E}_3)\psi_{\theta,\theta}, \end{aligned} \quad (5b)$$

$$\begin{aligned} N_{r\theta} = & \frac{\bar{G}_0}{r}u_{0,\theta} - \frac{\bar{G}_0}{r}v_0 + \bar{G}_0v_{0,r} + \frac{2C_1\bar{G}_3}{r^2}w_{,\theta} - \frac{2C_1\bar{G}_3}{r}w_{,r\theta} \\ & + (\bar{G}_1 - C_1\bar{G}_3)\left(\frac{1}{r}(\psi_{r,\theta} - \psi_\theta) + \psi_{\theta,r}\right), \end{aligned} \quad (5c)$$

$$Q_r = (\bar{G}_0 - C_2\bar{G}_2)(\psi_r + w_{,r}), \quad (5d)$$

$$Q_\theta = (\bar{G}_0 - C_2\bar{G}_2)\left(\psi_\theta + \frac{1}{r}w_{,\theta}\right), \quad (5e)$$

$$R_r = (\bar{G}_2 - C_2\bar{G}_4)(\psi_r + w_{,r}), \quad (5f)$$

$$R_\theta = (\bar{G}_2 - C_2\bar{G}_4)\left(\psi_\theta + \frac{1}{r}w_{,\theta}\right), \quad (5g)$$

$$\begin{aligned} M_r = & \frac{\bar{D}_1}{r}u_0 + \bar{E}_1u_{0,r} + \frac{\bar{D}_1}{r}v_{0,\theta} - \frac{C_1\bar{D}_4}{r}w_{,r} - C_1\bar{E}_4w_{,rr} - \frac{C_1\bar{D}_4}{r^2}w_{,\theta\theta} \\ & + \frac{1}{r}(\bar{D}_2 - C_1\bar{D}_4)\psi_r + (\bar{E}_2 - C_1\bar{E}_4)\psi_{r,r} + \frac{1}{r}(\bar{D}_2 - C_1\bar{D}_4)\psi_{\theta,\theta}, \end{aligned} \quad (5h)$$

$$M_\theta = \frac{\bar{E}_1}{r}u_0 + \bar{D}_1u_{0,r} + \frac{\bar{E}_1}{r}v_{0,\theta} - \frac{C_1\bar{E}_4}{r}w_{,r} - C_1\bar{D}_4w_{,rr} - \frac{C_1\bar{E}_4}{r^2}w_{,\theta\theta}$$

$$+ \frac{1}{r}(\bar{E}_2 - C_1\bar{E}_4)\psi_r + (\bar{D}_2 - C_1\bar{D}_4)\psi_{r,r} + \frac{1}{r}(\bar{E}_2 - C_1\bar{E}_4)\psi_{\theta,\theta} , \quad (5i)$$

$$M_{r\theta} = \frac{\bar{G}_1}{r}u_{0,\theta} - \frac{\bar{G}_1}{r}v_0 + \bar{G}_1v_{0,r} + \frac{2C_1\bar{G}_4}{r^2}w_{,\theta} - \frac{2C_1\bar{G}_4}{r}w_{,r\theta} \\ + (\bar{G}_2 - C_1\bar{G}_4)\left(\frac{1}{r}(\psi_{r,\theta} - \psi_\theta) + \psi_{\theta,r}\right) , \quad (5j)$$

$$P_r = \frac{\bar{D}_3}{r}u_0 + \bar{E}_3u_{0,r} + \frac{\bar{D}_3}{r}v_{0,\theta} - \frac{C_1\bar{D}_6}{r}w_{,r} - C_1\bar{E}_4w_{,rr} - \frac{C_1\bar{D}_6}{r^2}w_{,\theta\theta} \\ + \frac{1}{r}(\bar{D}_4 - C_1\bar{D}_6)\psi_r + (\bar{E}_4 - C_1\bar{E}_6)\psi_{r,r} + \frac{1}{r}(\bar{D}_4 - C_1\bar{D}_6)\psi_{\theta,\theta} , \quad (5k)$$

$$P_\theta = \frac{\bar{E}_3}{r}u_0 + \bar{D}_3u_{0,r} + \frac{\bar{E}_3}{r}v_{0,\theta} - \frac{C_1\bar{E}_6}{r}w_{,r} - C_1\bar{D}_6w_{,rr} - \frac{C_1\bar{E}_6}{r^2}w_{,\theta\theta} \\ + \frac{1}{r}(\bar{E}_4 - C_1\bar{E}_6)\psi_r + (\bar{D}_4 - C_1\bar{D}_6)\psi_{r,r} + \frac{1}{r}(\bar{E}_4 - C_1\bar{E}_6)\psi_{\theta,\theta} , \quad (5l)$$

$$P_{r\theta} = \frac{\bar{G}_3}{r}u_{0,\theta} - \frac{\bar{G}_3}{r}v_0 + \bar{G}_3v_{0,r} + \frac{2C_1\bar{G}_6}{r^2}w_{,\theta} - \frac{2C_1\bar{G}_6}{r}w_{,r\theta} \\ + (\bar{G}_4 - C_1\bar{G}_6)\left(\frac{1}{r}(\psi_{r,\theta} - \psi_\theta) + \psi_{\theta,r}\right) , \quad (5m)$$

where

$$\bar{G}_i = \int_{-h/2}^{h/2} Gz^i dz , \quad \bar{E}_i = \int_{-h/2}^{h/2} \frac{E}{1-\nu^2} z^i dz , \quad \bar{D}_i = \int_{-h/2}^{h/2} \frac{\nu E}{1-\nu^2} z^i dz , \quad (5n)$$

and G , E , and ν are shear modulus, modulus of elasticity, and Poisson's ratio, respectively, all of which can be functions of z . Since the variation range of Poisson's ratio is small and the stress singularity order at the interface corner in a bi-material isotropic plate is not sensitive to Poisson's ratio (Huang, 2002a), the Poisson's ratio is assumed constant through the thickness in the following. Then, \bar{D}_i , \bar{E}_i , and \bar{G}_i defined in Eq. (5n) satisfy the following relations:

$$\bar{D}_i = \nu\bar{E}_i \quad \text{and} \quad \bar{G}_i = \frac{(1-\nu)}{2}\bar{E}_i. \quad (6)$$

Substituting Eqs. (5a-5m) and (6) into Eqs. (3a-3e) yields the equilibrium equations for displacement functions as

$$\bar{E}_0\left(-\frac{u_0}{r^2} + \frac{u_{0,r}}{r} + u_{0,rr} + \frac{1-\nu}{2r^2}u_{0,\theta\theta} - \frac{3-\nu}{2r^2}v_{0,\theta} + \frac{1+\nu}{2r}v_{0,r\theta}\right) \\ + C_1\bar{E}_3\left(\frac{w_{,r}}{r^2} - \frac{w_{,rr}}{r} - w_{,rrr} + \frac{3+\nu}{2r^3}w_{,\theta\theta} - \frac{w_{,r\theta\theta}}{r^2}\right) + (\bar{E}_1 - C_1\bar{E}_3)\left(\frac{\psi_r}{r^2} + \frac{\psi_{r,r}}{r} + \psi_{r,rr}\right. \\ \left. + \frac{1-\nu}{2r^2}\psi_{r,\theta\theta} - \frac{3-\nu}{2r^2}\psi_{\theta,\theta} + \frac{1+\nu}{2r}\psi_{\theta,r\theta}\right) = 0 , \quad (7a)$$

$$\bar{E}_0\left(\frac{3-\nu}{2r^2}u_{0,\theta} + \frac{1+\nu}{2r}u_{0,r\theta} - \frac{1-\nu}{2r^2}v_0 + \frac{1-\nu}{2r}v_{0,r} + \frac{1-\nu}{2}v_{0,rr} + \frac{v_{0,\theta\theta}}{r^2}\right) \\ + C_1\bar{E}_3\left(-\frac{w_{,r\theta}}{r^2} - \frac{1+\nu}{2r}w_{,rr\theta} - \frac{w_{,\theta\theta\theta}}{r^3}\right) + (\bar{E}_1 - C_1\bar{E}_3)\left(\frac{3-\nu}{2r^2}\psi_{r,\theta} + \frac{1+\nu}{2r}\psi_{r,r\theta}\right)$$

$$-\frac{1-\nu}{2r^2}\psi_\theta + \frac{1-\nu}{2r}\psi_{\theta,r} + \frac{1-\nu}{2}\psi_{\theta,rr} + \frac{\psi_{\theta,\theta\theta}}{r^2} = 0 \quad , \quad (7b)$$

$$\begin{aligned} C_1\bar{E}_3 & \left(\frac{u_0 + v_{0,\theta}}{r^3} - \frac{u_{0,r} + v_{0,r\theta}}{r^2} + \frac{2u_{0,rr} + v_{0,rr\theta}}{r} + u_{0,rrr} + \frac{u_{0,\theta\theta} + v_{0,\theta\theta\theta}}{r^3} + \frac{u_{0,r\theta\theta}}{r^2} \right) \\ & + C_1^2\bar{E}_6 \left(-\frac{w_{,r}}{r^3} + \frac{w_{,rr}}{r^2} - \frac{4w_{,\theta\theta}}{r^4} + \frac{2w_{,r\theta\theta}}{r^3} - \frac{2w_{,rrr}}{r} - \frac{w_{,\theta\theta\theta}}{r^4} - \frac{2w_{,rr\theta\theta}}{r^2} - w_{,rrrr} \right) \\ & + \frac{1-\nu}{2}(\bar{E}_0 - 2C_2\bar{E}_2 + C_2^2\bar{E}_4) \left(\frac{w_{,r}}{r} + w_{,rr} + \frac{\psi_r}{r} + \psi_{r,r} + \frac{\psi_{\theta,\theta}}{r} \right) \\ & + (C_1\bar{E}_4 - C_1^2\bar{E}_6) \left(\frac{\psi_r + \psi_{\theta,\theta}}{r^3} - \frac{\psi_{r,r} + \psi_{\theta,r\theta}}{r^2} + \frac{\psi_{r,\theta\theta} + \psi_{\theta,\theta\theta\theta}}{r^3} \right. \\ & \left. + \frac{2\psi_{r,rr} + \psi_{\theta,rr\theta}}{r} + \frac{\psi_{r,r\theta\theta}}{r^2} + \psi_{r,rrr} \right) = 0 \quad , \quad (7c) \end{aligned}$$

$$\begin{aligned} (\bar{E}_1 - C_1\bar{E}_3) & \left(-\frac{u_0}{r^2} + \frac{u_{0,r}}{r} + u_{0,rr} + \frac{1-\nu}{2r^2}u_{0,\theta\theta} - \frac{3-\nu}{2r^2}v_{0,\theta} + \frac{1+\nu}{2r}v_{0,r\theta} \right) \\ & + (C_1\bar{E}_4 - C_1^2\bar{E}_6) \left(\frac{w_{,r}}{r^2} - \frac{w_{,rr}}{r} + \frac{2w_{,\theta\theta}}{r^3} - w_{,rrr} - \frac{w_{,r\theta\theta}}{r^2} \right) \\ & - \frac{1-\nu}{2}(\bar{E}_0 - 2C_2\bar{E}_2 + C_2^2\bar{E}_4)(w_{,r} + \psi_r) + \\ & (\bar{E}_2 - 2C_1\bar{E}_4 + C_1^2\bar{E}_6) \left(-\frac{\psi_r}{r^2} + \frac{\psi_{r,r}}{r} + \frac{1-\nu}{2r^2}\psi_{r,\theta\theta} + \psi_{r,rr} - \frac{3-\nu}{2r^2}\psi_{\theta,\theta} + \frac{1+\nu}{2r}\psi_{\theta,r\theta} \right) = 0 \quad , \quad (7d) \end{aligned}$$

$$\begin{aligned} (\bar{E}_1 - C_1\bar{E}_3) & \left(\frac{3-\nu}{2r^2}u_{0,\theta} + \frac{1+\nu}{2r}u_{0,r\theta} - \frac{1-\nu}{2}\left(\frac{v_0}{r^2} - \frac{v_{0,r}}{r} - v_{0,rr}\right) + \frac{v_{0,\theta\theta}}{r^2} \right) \\ & - (C_1\bar{E}_4 - C_1^2\bar{E}_6) \left(\frac{w_{,r\theta}}{r^2} + \frac{w_{,\theta\theta\theta}}{r^3} + \frac{w_{,rr\theta}}{r} \right) - \frac{1-\nu}{2}(\bar{E}_0 - 2C_2\bar{E}_2 + C_2^2\bar{E}_4) \left(\frac{w_{,\theta}}{r} + \psi_\theta \right) \\ & + (\bar{E}_2 - 2C_1\bar{E}_4 + C_1^2\bar{E}_6) \left(\frac{3-\nu}{2r^2}\psi_{r,\theta} + \frac{1+\nu}{2r}\psi_{r,r\theta} - \frac{1-\nu}{2r^2}\psi_\theta \right. \\ & \left. + \frac{1-\nu}{2r}\psi_{\theta,r} + \frac{\psi_{\theta,\theta\theta}}{r^2} + \frac{1-\nu}{2}\psi_{\theta,rr} \right) = 0 \quad , \quad (7e) \end{aligned}$$

Apparently, the in-plane displacement components u_0 and v_0 are coupled with out-of-plane displacement w in the mid-plane.

3. Asymptotic solutions

The eigenfunction expansion approach proposed by Hartranft and Sih (1969) for three-dimensional elasticity problems is adopted herein to find the solution of Eqs. (7a-7e). The displacement components can be expressed in terms of the following series

$$\begin{aligned} u_0(r, \theta) &= \sum_{i=0}^{\infty} \sum_{n=0}^{\infty} r^{\lambda_i+n} U_n^{(i)}(\theta, \lambda_i) \quad , \quad v_0(r, \theta) = \sum_{i=0}^{\infty} \sum_{n=0}^{\infty} r^{\lambda_i+n} V_n^{(i)}(\theta, \lambda_i) \quad , \\ w(r, \theta) &= \sum_{i=0}^{\infty} \sum_{n=0}^{\infty} r^{\lambda_i+n+1} W_n^{(i)}(\theta, \lambda_i) \quad , \quad \psi_r(r, \theta) = \sum_{i=0}^{\infty} \sum_{n=0}^{\infty} r^{\lambda_i+n} \Psi_n^{(i)}(\theta, \lambda_i) \quad , \end{aligned}$$

$$\psi_\theta(r, \theta) = \sum_{i=0}^{\infty} \sum_{n=0}^{\infty} r^{\lambda_i+n} \Phi_n^{(i)}(\theta, \lambda_i), \quad (8)$$

where the characteristic values λ_i are assumed to be constants and can be complex numbers.

The real part of λ_i must exceed zero to satisfy the regularity conditions at the vertex of the sector plate. The regularity conditions require that u_0 , v_0 , ψ_θ , ψ_r , w and $w_{,r}$ are finite as r approaches zero. As a result, the solution form given in Eq. (8) with the real part of λ_i less than one leads to singularities of N_r , N_θ , $N_{r\theta}$, M_r , M_θ , $M_{r\theta}$, P_r , P_θ and $P_{r\theta}$, which is observed from the relationships between stress resultants and displacement components given in Eqs. (5a-5m). However, no singularity for shear forces (Q_r and Q_θ), R_r and R_θ will be produced from the solution.

Substituting Eq. (8) into Eqs. (7a-7e) and considering those solutions corresponding to the lowest order of r , which dominate the stress singularity features at the neighborhood of $r=0$, yield

$$\begin{aligned} & \bar{E}_0 \left[-1 + \lambda_i + \lambda_i(\lambda_i - 1) \right] U_0^{(i)} + \frac{1-\nu}{2} \bar{E}_0 U_{0,\theta\theta}^{(i)} + \left(-\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) \bar{E}_0 V_{0,\theta}^{(i)} \\ & + C_1 \bar{E}_3 [\lambda_i + 1 - \lambda_i(\lambda_i + 1) - (\lambda_i + 1)\lambda_i(\lambda_i - 1)] W_0^{(i)} + C_1 \bar{E}_3 [2 - (\lambda_i + 1)] W_{0,\theta\theta}^{(i)} \\ & + (\bar{E}_1 - C_1 \bar{E}_3) [\lambda_i - 1 + \lambda_i(\lambda_i - 1)] \Psi_0^{(i)} + (\bar{E}_1 - C_1 \bar{E}_3) \frac{1-\nu}{2} \Psi_{0,\theta\theta}^{(i)} \\ & + (\bar{E}_1 - C_1 \bar{E}_3) \left(-\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) \Phi_{0,\theta}^{(i)} = 0, \end{aligned} \quad (9a)$$

$$\begin{aligned} & \bar{E}_0 \left(\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) U_{0,\theta}^{(i)} + \bar{E}_0 \frac{1-\nu}{2} [\lambda_i - 1 + \lambda_i(\lambda_i - 1)] V_0^{(i)} + \bar{E}_0 V_{0,\theta\theta}^{(i)} \\ & + C_1 \bar{E}_3 [-(\lambda_i + 1) - 2\lambda_i(\lambda_i + 1)] W_{0,\theta}^{(i)} - C_1 \bar{E}_3 W_{0,\theta\theta\theta}^{(i)} + (\bar{E}_1 - C_1 \bar{E}_3) \left(\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) \Psi_{0,\theta}^{(i)} \\ & + (\bar{E}_1 - C_1 \bar{E}_3) \frac{1-\nu}{2} [\lambda_i - 1 + \lambda_i(\lambda_i - 1)] \Phi_0^{(i)} + (\bar{E}_1 - C_1 \bar{E}_3) \Phi_{0,\theta\theta}^{(i)} = 0, \end{aligned} \quad (9b)$$

$$\begin{aligned} & C_1 \bar{E}_3 (\lambda_i - 1)^2 (\lambda_i + 1) U_0^{(i)} + C_1 \bar{E}_3 (\lambda_i + 1) U_{0,\theta\theta}^{(i)} - C_1^2 \bar{E}_6 (\lambda_i + 1)^2 (\lambda_i - 1)^2 W_0^{(i)} \\ & + C_1^2 \bar{E}_6 [-4 + 2(\lambda_i + 1) - 2\lambda_i(\lambda_i + 1)] W_{0,\theta\theta}^{(i)} - C_1^2 \bar{E}_6 W_{0,\theta\theta\theta\theta}^{(i)} \\ & + (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (\lambda_i - 1)^2 (\lambda_i + 1) \Psi_0^{(i)} + (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (\lambda_i + 1) \Psi_{0,\theta\theta}^{(i)} \\ & + (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (\lambda_i - 1)^2 \Phi_{0,\theta}^{(i)} + (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) \Phi_{0,\theta\theta\theta}^{(i)} = 0, \end{aligned} \quad (9c)$$

$$\begin{aligned} & (\bar{E}_1 - C_1 \bar{E}_3) (\lambda_i - 1) (\lambda_i + 1) U_0^{(i)} + \frac{1-\nu}{2} (\bar{E}_1 - C_1 \bar{E}_3) U_{0,\theta\theta}^{(i)} \\ & + (\bar{E}_1 - C_1 \bar{E}_3) \left(-\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) V_{0,\theta}^{(i)} - (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (\lambda_i + 1)^2 (\lambda_i - 1) W_0^{(i)} \\ & + (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (1 - \lambda_i) W_{0,\theta\theta}^{(i)} + (\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6) (\lambda_i + 1) (\lambda_i - 1) \Psi_0^{(i)} \\ & + (\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6) \frac{1-\nu}{2} \Psi_{0,\theta\theta}^{(i)} + \left(-\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) \Phi_{0,\theta}^{(i)} = 0, \end{aligned} \quad (9d)$$

$$(\bar{E}_1 - C_1 \bar{E}_3) \left(\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) U_{0,\theta}^{(i)} + (\bar{E}_1 - C_1 \bar{E}_3) \frac{1-\nu}{2} (\lambda_i + 1) (\lambda_i - 1) V_0^{(i)}$$

$$\begin{aligned}
& +(\bar{E}_1 - C_1 \bar{E}_3) V_{0,\theta\theta}^{(i)} - (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) (\lambda_i + 1)^2 W_{0,\theta}^{(i)} - (C_1 \bar{E}_4 - C_1^2 \bar{E}_6) W_{0,\theta\theta}^{(i)} \\
& + \left(\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6 \right) \left(\frac{3-\nu}{2} + \frac{1+\nu}{2} \lambda_i \right) \Psi_{0,\theta}^{(i)} \\
& + \left(\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6 \right) \left[\left(-\frac{1-\nu}{2} + \frac{1-\nu}{2} \lambda_i^2 \right) \Phi_0^{(i)} + \Phi_{0,\theta\theta}^{(i)} \right] = 0.
\end{aligned} \tag{9e}$$

Equations (9) are a set of linear differential equations with constant coefficients. Following a typical procedure for solving a set of linear differential equations, the general solutions for $U_0^{(i)}$, $V_0^{(i)}$ and $W_0^{(i)}$ are obtained as

$$U_0^{(i)}(\theta, \lambda_i) = A_1 \cos(\lambda_i + 1)\theta + A_2 \sin(\lambda_i + 1)\theta + A_3 \cos(\lambda_i - 1)\theta + A_4 \sin(\lambda_i - 1)\theta, \tag{10a}$$

$$\begin{aligned}
V_0^{(i)}(\theta, \lambda_i) = & A_2 \cos(\lambda_i + 1)\theta - A_1 \sin(\lambda_i + 1)\theta + (\kappa_1 A_4 - \kappa_2 B_4) \cos(\lambda_i - 1)\theta \\
& + (-\kappa_1 A_3 + \kappa_2 B_3) \sin(\lambda_i - 1)\theta,
\end{aligned} \tag{10b}$$

$$W_0^{(i)}(\theta, \lambda_i) = B_1 \cos(\lambda_i + 1)\theta + B_2 \sin(\lambda_i + 1)\theta + B_3 \cos(\lambda_i - 1)\theta + B_4 \sin(\lambda_i - 1)\theta, \tag{10c}$$

$$\Psi_0^{(i)}(\theta, \lambda_i) = D_1 \cos(\lambda_i + 1)\theta + D_2 \sin(\lambda_i + 1)\theta + D_3 \cos(\lambda_i - 1)\theta + D_4 \sin(\lambda_i - 1)\theta, \tag{10d}$$

$$\begin{aligned}
\Phi_0^{(i)}(\theta, \lambda_i) = & D_2 \cos(\lambda_i + 1)\theta - D_1 \sin(\lambda_i + 1)\theta + (\kappa_1 D_4 - \kappa_3 B_4) \cos(\lambda_i - 1)\theta \\
& + (-\kappa_1 D_3 + \kappa_3 B_3) \sin(\lambda_i - 1)\theta,
\end{aligned} \tag{10e}$$

where $\kappa_1 = \frac{3 + \lambda_i - \nu + \lambda_i \nu}{-3 + \lambda_i + \nu + \lambda_i \nu}$,

$$\kappa_2 = - \frac{8C_1 \lambda_i (\bar{E}_2 \bar{E}_3 - C_1 \bar{E}_3 \bar{E}_4 + \bar{E}_1 (-\bar{E}_4 + C_1 \bar{E}_6))}{\bar{E}_1^2 - 2C_1 \bar{E}_1 \bar{E}_3 + C_1^2 \bar{E}_3^2 - \bar{E}_0 (\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6)} \quad \text{and}$$

$$\kappa_3 = \frac{8C_1 \lambda_i (\bar{E}_1 \bar{E}_3 - C_1 \bar{E}_3^2 + \bar{E}_0 (-\bar{E}_4 + C_1 \bar{E}_6))}{\bar{E}_1^2 - 2C_1 \bar{E}_1 \bar{E}_3 + C_1^2 \bar{E}_3^2 - \bar{E}_0 (\bar{E}_2 - 2C_1 \bar{E}_4 + C_1^2 \bar{E}_6)}.$$

Coefficients A_i , B_i and D_i ($i=1, 2, 3, 4$) and λ_i are to be determined from boundary conditions along the radial edges. Notably, the extension and bending of a plate will be independent when \bar{E}_1 and \bar{E}_3 equal zero.

4. Boundary conditions, characteristic equations and corner functions

After solving the equilibrium equations, attention is now turned to find out A_i , B_i , D_i and λ_i in the solutions from the boundary conditions along the radial edges forming a corner. To determine Williams type stress singularities at the vertex of a sector plate caused by homogeneous boundary conditions, one only needs the asymptotic solution with the lowest order of r in the series solution of Eq.(8). Consequently, only the solution with $n=0$ in Eq. (8) needs to be considered. Let,

$$\begin{aligned}
u_0^{(m)} &= r^{\lambda_m} U_0^{(m)}(\theta, \lambda_m), \quad v_0^{(m)} = r^{\lambda_m} V_0^{(m)}(\theta, \lambda_m) \\
\psi_{\theta 0}^{(m)} &= r^{\lambda_m} \Phi_0^{(m)}(\theta, \lambda_m), \quad \psi_{r 0}^{(m)} = r^{\lambda_m} \Psi_0^{(m)}(\theta, \lambda_m), \quad \text{and} \quad w_0^{(m)} = r^{\lambda_m + 1} W_0^{(m)}(\theta, \lambda_m)
\end{aligned} \tag{11}$$

Furthermore, as well known, the stress singularities are affected by the boundary conditions along radial edges only.

In the following, we will consider four types of homogeneous boundary conditions along a

radial edge, say $\theta = \alpha$, namely,

$$\text{clamped: } u_0 = v_0 = w = \psi_r = \psi_\theta = \frac{w_{,\theta}}{r} = 0, \quad (12a)$$

$$\text{free: } N_{r\theta} = N_\theta = \bar{M}_\theta = \bar{M}_{r\theta} = \bar{Q}_\theta + C_1\left(\frac{2}{r}P_{r\theta} + 2P_{r\theta,r} + \frac{1}{r}P_{\theta,\theta}\right) = P_\theta = 0, \quad (12b)$$

$$\text{type I simply supported: } u_0 = v_0 = w = \psi_r = \bar{M}_\theta = P_\theta = 0, \quad (12c)$$

$$\text{type II simply supported: } u_0 = v_0 = w = \bar{M}_\theta = \bar{M}_{r\theta} = P_\theta = 0. \quad (12d)$$

For the simplicity, C and F are used to present the clamped and free boundary conditions, respectively, while S(I) and S(II) denote type I and type II simply supported boundary conditions.

For the sake of demonstration, we will describe the procedure for obtaining the characteristic equation for λ_m , and the corresponding asymptotic displacement field for describing the singular behavior of stress resultants in the vicinity of a corner. A wedge with simply supported radial edges and a vertex angle α is utilized to demonstrate a typical procedure for deriving these coefficients and λ . S(I) boundary conditions simulate the mechanical support of a knife-edge along the mid-plane. By taking advantage of the problem's symmetry, the solutions given in Eq. (11) are separated into symmetric and anti-symmetric parts. The symmetric solutions satisfying the boundary conditions yield

$$\begin{aligned} & A_1 \cos(\lambda_m + 1) \frac{\alpha}{2} + A_3 \cos(\lambda_m - 1) \frac{\alpha}{2} = 0, \\ & -A_1 \sin(\lambda_m + 1) \frac{\alpha}{2} + (-\kappa_1 A_3 + \kappa_2 B_3) \sin(\lambda_m - 1) \frac{\alpha}{2} = 0, \\ & B_1 \cos(\lambda_m + 1) \frac{\alpha}{2} + B_3 \cos(\lambda_m - 1) \frac{\alpha}{2} = 0, \quad D_1 \cos(\lambda_m + 1) \frac{\alpha}{2} + D_3 \cos(\lambda_m - 1) \frac{\alpha}{2} = 0, \\ & (\lambda_m + 1) \{-A_1 \bar{E}_1 + D_1 (-\bar{E}_2 + C_1 \bar{E}_4) + B_1 C_1 \bar{E}_4 (\lambda_m + 1)\} \cos(\lambda_m + 1) \frac{\alpha}{2} \\ & + (\lambda_m - 1) \{-A_3 \kappa_1 \bar{E}_1 + \kappa_1 (-\bar{E}_2 + C_1 \bar{E}_4) D_3 + (\kappa_2 \bar{E}_1 + \kappa_3 \bar{E}_4 - C_1 \bar{E}_4 (1 + \kappa_3 - \lambda_m) B_3\} \cos(\lambda_m - 1) \frac{\alpha}{2} = 0 \\ & (\lambda_m + 1) \{-A_1 \bar{E}_3 + D_1 (-\bar{E}_4 + C_1 \bar{E}_6) + B_1 C_1 \bar{E}_6 (\lambda_m + 1)\} \cos(\lambda_m + 1) \frac{\alpha}{2} \\ & + (\lambda_m - 1) \{-A_3 \kappa_1 \bar{E}_1 + \kappa_1 (-\bar{E}_4 + C_1 \bar{E}_6) D_3 + (\kappa_2 \bar{E}_3 + \kappa_3 \bar{E}_4 - C_1 \bar{E}_6 (1 + \kappa_3 - \lambda_m) B_3\} \cos(\lambda_m - 1) \frac{\alpha}{2} = 0 \end{aligned} \quad (13)$$

Equations (13) are a set of six linear algebraic equations for A_i , B_i and D_i ($i=1$ and 3). To have nontrivial solutions for these coefficients yields a 6th order determinant equal to zero, which leads to

$$\cos(\lambda_m + 1) \frac{\alpha}{2} = 0, \quad \cos(\lambda_m - 1) \frac{\alpha}{2} = 0, \quad (14a)$$

$$\text{or } \bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0 \quad (14b)$$

where $\bar{\gamma}_1 = \frac{\bar{\kappa}_1}{\bar{\kappa}_2}$, $\bar{\kappa}_1 = [-C_1 \bar{E}_4 (-\kappa_2 \bar{E}_3 + 2(1 + \kappa_1) \bar{E}_4) + \kappa_2 \bar{E}_1 (\bar{E}_4 - C_1 \bar{E}_6) + \bar{E}_2 (-\kappa_2 \bar{E}_3 + 2C_1 (1 + \kappa_1) \bar{E}_6)]$,

$$\bar{\kappa}_2 = [C_1 \bar{E}_4 (-\kappa_2 \bar{E}_3 + 2(-1 + \kappa_1) \lambda_m \bar{E}_4) - \kappa_2 \bar{E}_1 (\bar{E}_4 - C_1 \bar{E}_6) - \bar{E}_2 (-\kappa_2 \bar{E}_3 + 2C_1 (-1 + \kappa_1) \lambda_m \bar{E}_6)].$$

When λ_m satisfies $\cos(\lambda_m + 1) \frac{\alpha}{2} = 0$ and $\cos(\lambda_m - 1) \frac{\alpha}{2} \neq 0$, the relations among A_i , B_i and

D_i ($i=1$ and 3) are obtained from Eq. (13). Then, the corresponding asymptotic solutions are $w_0^{(m)} = B_1 r^{\lambda_m+1} \{\cos(\lambda_m+1)\theta\}$, $\psi_{r0}^{(m)} = D_1 r^{\lambda_m} \{\cos(\lambda_m+1)\theta\}$, $\psi_{\theta 0}^{(m)} = D_1 r^{\lambda_m} \{-\sin(\lambda_m+1)\theta\}$. (15)

The asymptotic solutions for $u_0^{(m)}$ and $v_0^{(m)}$ have the order of r larger than λ_m and do not cause stress singularities. These asymptotic solutions given in Eq. (15) are also called corner functions corresponding to the characteristic equation $\cos(\lambda_m+1)\frac{\alpha}{2} = 0$. The asymptotic solutions for in-plane and out-of-plane displacement components on the mid-plane are independent.

Similarly, one can find the corner functions corresponding to different characteristic equations (i.e., $\cos(\lambda_m-1)\frac{\alpha}{2} = 0$ or $\bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0$ in Eqs. (14)), which are listed in Table 1.

Notably, $\cos(\lambda_m-1)\frac{\alpha}{2} = 0$ and $\cos(\lambda_m+1)\frac{\alpha}{2} \neq 0$ yields the asymptotic solution for $v_0^{(m)}$ with the order of r larger than λ_m , which is not given in Table 1.

Using the anti-symmetric parts of solutions in Eq. (11) and following the procedure above, the following characteristic equations are easily established:

$$(\sin(\lambda_m+1)\frac{\alpha}{2})(\sin(\lambda_m-1)\frac{\alpha}{2}) = 0, \quad (16a)$$

or

$$\sin \lambda_m \alpha - \bar{\gamma}_1 \lambda_m \sin \alpha = 0. \quad (16b)$$

The corner functions corresponding to these characteristic equations are also given in Table 1.

Notably, $(\sin(\lambda_m+1)\frac{\alpha}{2}) = 0$ and $(\sin(\lambda_m-1)\frac{\alpha}{2}) \neq 0$ also yield the asymptotic solutions for $u_0^{(m)}$ and $v_0^{(m)}$ having the order of r larger than λ_m , while $(\sin(\lambda_m-1)\frac{\alpha}{2}) = 0$ and $(\sin(\lambda_m+1)\frac{\alpha}{2}) \neq 0$ gives the asymptotic solution for $v_0^{(m)}$ with the order of r larger than λ_m .

These asymptotic solutions do not yield stress singularities and are not listed in Table 1.

Characteristic Eqs.(14a) and (16a) do not involve material properties, and are identical to the characteristic equations for a homogeneous thick sectorial plate with two simply supported radial edges under bending. However, material properties are involved in the characteristic equations Eqs.(14b) and (16b).

By following the procedure given above, one can develop the characteristic equations for λ_m and the corresponding corner functions for different boundary conditions along radial edges. Table 2 summarizes the characteristic equations for λ_m for eight different combinations of boundary conditions. To take advantage of the problem's symmetry, the corner functions for the identical boundary conditions along two radial edges were determined by considering the range, $-\alpha/2 \leq \theta \leq \alpha/2$. Table 1 summarizes the corner functions for S(I)_S(I), S(II)_S(II), C_C and F-F, which are already considerably complicated. The corner functions corresponding to other boundary conditions are too lengthy and complicated to be given in Table 1. These results given in Tables 1 and 2 are the first shown in the published literature.

Notably, Table 2 does not list the characteristic equations for S(I)_F and S(II)_F boundary conditions because they are much more complicated and lengthy than those given in Table 2. To find the values of λ_m for these boundary conditions, it is suggested to determine them directly from solving the zeros of the 12th order determinant established from S(I)_F or S(II)_F boundary conditions.

Table 2 reveals that the characteristic equations corresponding to the boundary conditions

without involving S(I) or S(II) condition are identical to the combination of the characteristic equations for homogeneous plates under bending and extension (Williams (1952a), Huang (2002)). The non-homogeneity considered here does not influence the stress singularity orders at the corner of a plate if one of the two radial edges around the corner is not simply supported. However, Table 1 shows that most of the asymptotic solutions for the in-plane and out-of-plane displacement components on the mid-plane are coupled and are significantly different from those for homogenous plates.

5. Numerical results for λ_m

To demonstrate the effects of material non-homogeneity on stress singularity orders, a typical non-homogeneity is considered, assuming the variations of the Young's modulus through the thickness of plate given as

$$E(z) = E_b + V(z)\Delta E, \quad (17)$$

where $V(z) = (z/h + 1/2)^m$. Consequently, the roots of the characteristic equations corresponding to boundary conditions involving simple support (see Table 2) depend on m , ν and $\Delta E/E_b$. The Poisson's ratio is set equal to 0.3 for the results shown below.

Figure 2 depicts the variation of the minimum real parts of λ_m ($\text{Re}[\lambda]$, the subscript m is omitted for simplicity) with the material properties and the vertex angle of a wedge having S(I)_S(I) boundary conditions along radial edges. The roots of Eqs. (14a), (14b), (16a), and (16b) were obtained independently. An infinite number of roots exist for each of these equations. Only the root with a minimum real part is important as it determines the stress singularity order at the vertex. The minimum values of $\text{Re}[\lambda]$ obtained from Eq. (14a), independent of material properties, are smaller than those for Eq. (14b) for a wide range of material properties when minimum $\text{Re}[\lambda]$ is less than unity. Similarly, minimum values of $\text{Re}[\lambda]$ determined from Eq.(16a) are less than those for Eq. (16b) with different material properties. As a result, the singularity order of the stress resultants at a sharp corner with S(I)_S(I) edges is very likely determined by Eq. (14a) or (16a), independent of material properties.

Figures 3 and 4 illustrate the effects of material non-homogeneity on the minimum values of $\text{Re}[\lambda]$ of the characteristic equations corresponding to S(I)_S(II) and S(I)_C boundary conditions, respectively. Since the characteristic equation involving material non-homogeneity for S(II)_C boundary conditions is the same as that for S(I)_C boundary conditions, Fig. 4 also illustrates the minimum values of $\text{Re}[\lambda]$ of the characteristic equations for S(II)_C boundary conditions. Table 2 reveals that characteristic equations without involving material non-homogeneity were found for S(I)_S(II), S(I)_C and S(II)_C boundary conditions. These characteristic equations are also found for homogeneous plates under bending (Huang (2002)). The results denoted as "homogenous" in Figs. 3 and 4 are the minimum values of $\text{Re}[\lambda]$ for these characteristic equations without involving material non-homogeneity. The material non-homogeneity considered in these figures does not significantly affect the stress singularity order at the vertex. The stress singularity order is mainly determined by the roots of the characteristic equations corresponding to a homogeneous plate under bending.

Figures 5 and 6 demonstrate the effects of material non-homogeneity on the minimum values of $\text{Re}[\lambda]$ of the characteristic equations corresponding to S(I)_F and S(II)_F boundary conditions, respectively. The roots of λ_m were determined from the zeros of the 12th order determinant established based on S(I)_F or S(II)_F boundary conditions. It was revealed that no root of λ_m is identical to that obtained from the characteristic equations for a homogenous. Figures 5 and 6 explore that material non-homogeneity significantly affects the stress singularity order at a corner with S(I)_F or S(II)_F boundary conditions along the two edges forming the corner.

6. Concluding remarks

This study has developed the equilibrium equations in terms of displacement functions for an FGM thick plate based on Reddy's plate theory and established the asymptotic displacement field to describe the singular behaviors of stress resultants in the vicinity of a sharp corner. The asymptotic solutions were obtained using the eigenfunction expansion method and assuming non-homogeneous Young's modulus and constant Poisson ratio along plate thickness. This work has also established the characteristic equations for determining stress singularity orders at the vertex of the corner with different boundary condition combinations. These asymptotic solutions and characteristic equations are the first given in the literature. Only the characteristic equations corresponding to boundary conditions involving simple support depend on material non-homogeneity. Nevertheless, regardless of the boundary conditions considered, the asymptotic solutions of in-plane and out-of-plane displacement components on the mid-plane are usually coupled.

In examining how material non-homogeneity affects stress singularity orders, this work considered the non-homogeneous Young's modulus following a power law. The present study considered S(I)_S(I), S(I)_S(II), S(I)_C, S(II)_C, S(I)_F and S(II)_F boundary conditions along the radial edges of a sector plate. The material non-homogeneity considered in the work does not significantly affect the stress singularity order at the vertex when S(I)_S(I), S(I)_S(II), S(I)_C, S(II)_C boundary conditions are considered. The stress singularity order is mainly determined by the roots of the characteristic equations corresponding to a homogeneous plate under bending. However, the material non-homogeneity does significantly affect the stress singularity order at a corner with S(I)_F or S(II)_F boundary conditions along the two edges forming the corner..

The corner functions shown here will be utilized in future FGM plate studies involving geometrically induced stress singularities to determine accurate free vibration frequencies and mode shapes of thick plates having such boundary discontinuities. Because other corner functions have been used advantageously for vibration studies of homogeneous plates (Huang *et al.*, 2005), the present corner functions are definitely appropriate for FGM thick plate vibration problems. These corner functions can also be used for static stress and deformation analysis, especially for determining the stress intensity factors for a V-notch or a crack.

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Table 1 Corner functions corresponding to symmetrical boundary conditions

Boundary conditions	Corner functions
<p>S(I)- S(I)</p> <p>$(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>(1) Symmetrical case</p> <p>When $\cos(\lambda_m - 1)\alpha/2 = 0$,</p> $u_0^{(m)} = A_3 r^{\lambda_m} \{\cos(\lambda_m - 1)\theta\}, \quad w_0^{(m)} = A_3 r^{\lambda_m+1} \left\{ \frac{\kappa_1}{\kappa_2} \cos(\lambda_m - 1)\theta \right\},$ $\psi_{r0}^{(m)} = r^{\lambda_m} \{D_3 \cos(\lambda_m - 1)\theta\}, \quad \psi_{\theta 0}^{(m)} = r^{\lambda_m} \left\{ \left(-\kappa_1 D_3 + \kappa_3 A_3 \frac{\kappa_1}{\kappa_2} \right) \sin(\lambda_m - 1)\theta \right\}.$ <p>When $\cos(\lambda_m + 1)\alpha/2 = 0$,</p> $w_0^{(m)} = B_1 r^{\lambda_m+1} \{\cos(\lambda_m + 1)\theta\}, \quad \psi_{r0}^{(m)} = D_1 r^{\lambda_m} \{\cos(\lambda_m + 1)\theta\},$ $\psi_{\theta 0}^{(m)} = D_1 r^{\lambda_m} \{-\sin(\lambda_m + 1)\theta\}.$ <p>When $\bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0$,</p> $u_0^{(m)} = A_3 r^{\lambda_m} \left\{ \left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + \cos(\lambda_m - 1)\theta \right\},$ $v_0^{(m)} = A_3 r^{\lambda_m} \left\{ \left(\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + (-\kappa_1 + \kappa_2 \bar{\eta}_2) \sin(\lambda_m - 1)\theta \right\},$ $w_0^{(m)} = A_3 r^{\lambda_m+1} \left\{ \bar{\eta}_2 \left[\left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + \cos(\lambda_m - 1)\theta \right] \right\},$ $\psi_{r0}^{(m)} = A_3 r^{\lambda_m} \left\{ \bar{\eta}_1 \left[\left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m - 1)\theta + \cos(\lambda_m + 1)\theta \right] \right\},$ $\psi_{\theta 0}^{(m)} = A_3 r^{\lambda_m} \left\{ \bar{\eta}_1 \left(\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + (-\kappa_1 \bar{\eta}_1 + \kappa_3 \bar{\eta}_2) \sin(\lambda_m - 1)\theta \right\},$

Table 1 (Continued)

<p>S(I)- S(I) $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>where</p> $\bar{\eta}_1 = \{[\kappa_3(\lambda_m - 1)\bar{E}_2 + C_1(\kappa_3 - 4\lambda_m - \kappa_3\lambda_m)\bar{E}_4][\kappa_1 - \cot[(\lambda_m - 1)\alpha/2]\tan[(\lambda_m + 1)\alpha/2]] \\ + \kappa_2\bar{E}_1[1 + \lambda_m - (\lambda_m - 1)\cot[(\lambda_m - 1)\alpha/2]\tan[(\lambda_m + 1)\alpha/2]]\} \\ \bigg/ [\kappa_2(-1 + \kappa_1(\lambda_m - 1) - \lambda_m)(\bar{E}_2 - C_1\bar{E}_4)],$ $\bar{\eta}_2 = \frac{1}{\kappa_2} \{\kappa_1 - \cot[(\lambda_m - 1)\alpha/2]\tan[(\lambda_m + 1)\alpha/2]\}.$ <p>(2) Anti-symmetrical case</p> <p>When $\sin(\lambda_m - 1)\alpha/2 = 0$,</p> $u_0^{(m)} = A_4 r^{\lambda_m} \{\sin(\lambda_m - 1)\theta\}, \quad w_0^{(m)} = A_4 r^{\lambda_m + 1} \left\{ \frac{\kappa_1}{\kappa_2} \sin(\lambda_m - 1)\theta \right\},$ $\psi_{r0}^{(m)} = r^{\lambda_m} \{D_4 \sin(\lambda_m - 1)\theta\}, \quad \psi_{\theta0}^{(m)} = r^{\lambda_m} \left\{ \left(\kappa_1 D_4 - \kappa_3 A_4 \frac{\kappa_1}{\kappa_2} \right) \cos(\lambda_m - 1)\theta \right\}.$ <p>When $\sin(\lambda_m + 1)\alpha/2 = 0$,</p> $w_0^{(m)} = B_2 r^{\lambda_m + 1} \{\sin(\lambda_m + 1)\theta\}, \quad \psi_{r0}^{(m)} = D_2 r^{\lambda_m} \{\sin(\lambda_m + 1)\theta\},$ $\psi_{\theta0}^{(m)} = D_2 r^{\lambda_m} \{\cos(\lambda_m + 1)\theta\}.$ <p>When $\sin \lambda_m \alpha - \bar{\gamma}_1 \lambda_m \sin \alpha = 0$,</p> $u_0^{(m)} = A_4 r^{\lambda_m} \left\{ \left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right\},$ $v_0^{(m)} = A_4 r^{\lambda_m} \left\{ \left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + (\kappa_1 - \kappa_2 \bar{\eta}_3) \cos(\lambda_m - 1)\theta \right\},$ $w_0^{(m)} = A_4 r^{\lambda_m + 1} \left\{ \bar{\eta}_3 \left[\left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right] \right\},$
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Table 1(Continued)

	$\psi_{r_0}^{(m)} = A_4 r^{\lambda_m} \left\{ \bar{\eta}_1 \left[\left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right] \right\},$ $\psi_{\theta_0}^{(m)} = A_4 r^{\lambda_m} \left\{ \bar{\eta}_1 \left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + (\kappa_1 \bar{\eta}_1 - \kappa_3 \bar{\eta}_3) \cos(\lambda_m - 1)\theta \right\},$ <p>where</p> $\bar{\eta}_3 = \frac{1}{\kappa_2} \{ \kappa_1 - \cot[(\lambda_m + 1)\alpha/2] \tan[(\lambda_m - 1)\alpha/2] \}.$
<p>S(II)-S(II)</p> <p>$(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>(1) Symmetrical case</p> <p>When $\cos(\lambda_m - 1)\frac{\alpha}{2} = 0$ and $\cos(\lambda_m + 1)\frac{\alpha}{2} = 0$, the corresponding corner functions are the same as those for S(I)-S(I).</p> <p>When $\bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0$ or $\lambda_m \sin \alpha + \sin \lambda_m \alpha = 0$,</p> $u_0^{(m)} = A_3 r^{\lambda_m} \left\{ \left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + \cos(\lambda_m - 1)\theta \right\},$ $v_0^{(m)} = A_3 r^{\lambda_m} \left\{ \left(\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + (-\kappa_1 + \kappa_2 \bar{\eta}_2) \sin(\lambda_m - 1)\theta \right\},$ $w_0^{(m)} = A_3 r^{\lambda_m + 1} \left\{ \bar{\eta}_2 \left[\left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + \cos(\lambda_m - 1)\theta \right] \right\},$ $\psi_{r_0}^{(m)} = A_3 r^{\lambda_m} \left\{ -\frac{1}{\lambda_m(\nu - 1)} \left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) (\bar{\eta}_4 - (1 + \kappa_1 - \kappa_1 \lambda_m + \lambda_m \nu) \bar{\eta}_5) \right.$ $\left. \cos(\lambda_m - 1)\theta - \bar{\eta}_5 \cos(\lambda_m + 1)\theta \right\},$ $\psi_{\theta_0}^{(m)} = A_3 r^{\lambda_m} \left\{ \frac{1}{\lambda_m(\nu - 1)} \left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) (\bar{\eta}_4 - (1 + \kappa_1 - \kappa_1 \lambda_m + \lambda_m \nu)) \right.$ $\left. \sin(\lambda_m + 1)\theta + (\kappa_1 \bar{\eta}_5 + \kappa_3 \bar{\eta}_2) \sin(\lambda_m - 1)\theta \right\},$ <p>where</p> $\bar{\eta}_4 = -\frac{\kappa_1(-1 + \lambda_m)\bar{E}_1}{\bar{E}_2 - C_1\bar{E}_4} + \frac{(1 + \lambda_m)\bar{E}_1}{\bar{E}_2 - C_1\bar{E}_4} + \frac{C_1(1 + \lambda_m)^2\bar{E}_4\bar{\eta}_2}{(-\bar{E}_2 + C_1\bar{E}_4)} +$ $\frac{\bar{\eta}_2}{(\bar{E}_2 - C_1\bar{E}_4)} (-1 + \lambda_m)[\kappa_2\bar{E}_1 + \kappa_3\bar{E}_2 + C_1(-1 - \kappa_3 + \lambda_m)\bar{E}_4],$

Table 1(Continued)

<p>S(II)-S(II) $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	$\begin{aligned} \bar{\eta}_5 = & \left\{ (-1 + \lambda_m)(-1 + \nu)(\bar{E}_1 - C_1\bar{E}_3)(\bar{E}_2 - C_1\bar{E}_4) \cos[(1 + \lambda_m)\frac{\alpha}{2}] \sin[(-1 + \lambda_m)\frac{\alpha}{2}] \right. \\ & + 2\kappa_1(-1 + \lambda_m)\bar{E}_1[\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)] \cos[(-1 + \lambda_m)\frac{\alpha}{2}] \sin[(1 + \lambda_m)\frac{\alpha}{2}] \\ & - (1 + \lambda_m)\{C_1(-1 + \nu)\bar{E}_3(-\bar{E}_2 + C_1\bar{E}_4) \\ & + \bar{E}_1[(1 + \nu)\bar{E}_2 + C_1(-3 + \nu)\bar{E}_4 + 2C_1\bar{E}_6]\} \cos[(-1 + \lambda_m)\frac{\alpha}{2}] \sin[(1 + \lambda_m)\frac{\alpha}{2}] \\ & - \bar{\eta}_2\{-2C_1(1 + \lambda_m)\{(-1 + \nu)(\bar{E}_2 - C_1\bar{E}_4)(\bar{E}_4 - C_1\bar{E}_6) \\ & - (1 + \lambda_m)\bar{E}_4[\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)]\} \cos[(-1 + \lambda_m)\frac{\alpha}{2}] \sin[(1 + \lambda_m)\frac{\alpha}{2}] \\ & + (-1 + \lambda_m)\{(-1 + \nu)(\bar{E}_2 - C_1\bar{E}_4) \\ & \{-\kappa_3\bar{E}_2 + C_1[2(1 + \kappa_3)\bar{E}_4 - C_1(2 + \kappa_3)\bar{E}_6]\} \cos[(1 + \lambda_m)\frac{\alpha}{2}] \sin[(-1 + \lambda_m)\frac{\alpha}{2}] \\ & - 2[\kappa_2\bar{E}_1 + \kappa_3\bar{E}_2 + C_1(-1 + \kappa_3 + \lambda_m)\bar{E}_4][\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)] \\ & \left. \cos[(-1 + \lambda_m)\frac{\alpha}{2}] \sin[(1 + \lambda_m)\frac{\alpha}{2}]\} \right\} / \{(\bar{E}_2 - C_1\bar{E}_4)(\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)) \\ & [(1 + \kappa_1)(-1 + \lambda_m)(-1 + \nu) \cos[(1 + \lambda_m)\frac{\alpha}{2}] \sin[(-1 + \lambda_m)\frac{\alpha}{2}] - \\ & 2(1 + \kappa_1 - \kappa_1\lambda_m + \lambda_m\nu) \cos[(-1 + \lambda_m)\frac{\alpha}{2}] \sin[(1 + \lambda_m)\frac{\alpha}{2}]]\}, \end{aligned}$ <p>(2) Anti-symmetrical case</p> <p>When $\sin(\lambda_m - 1)\frac{\alpha}{2} = 0$ and $\sin(\lambda_m + 1)\frac{\alpha}{2} = 0$, the corresponding corner functions are the same as those for S(I)- S(I).</p> <p>When $\lambda_m \sin \alpha - \sin \lambda_m \alpha = 0$ or $\sin \lambda_m \alpha - \bar{\gamma}_1 \lambda_m \sin \alpha = 0$,</p> $u_0^{(m)} = A_4 r^{\lambda_m} \left\{ -\frac{\sin(\lambda_m - 1)\frac{\alpha}{2}}{\sin(\lambda_m + 1)\frac{\alpha}{2}} \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right\},$ $v_0^{(m)} = A_4 r^{\lambda_m} \left\{ -\frac{\sin(\lambda_m - 1)\frac{\alpha}{2}}{\sin(\lambda_m + 1)\frac{\alpha}{2}} \cos(\lambda_m + 1)\theta + (\kappa_1 - \kappa_2\bar{\eta}_3) \cos(\lambda_m - 1)\theta \right\},$ $w_0^{(m)} = A_4 r^{\lambda_m + 1} \left\{ \bar{\eta}_3 \left[-\frac{\sin(\lambda_m - 1)\frac{\alpha}{2}}{\sin(\lambda_m + 1)\frac{\alpha}{2}} \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right] \right\},$ $\psi_{r_0}^{(m)} = A_4 r^{\lambda_m} \left\{ -\frac{1}{\lambda_m(\nu - 1)} \left(\frac{\sin(\lambda_m - 1)\frac{\alpha}{2}}{\sin(\lambda_m + 1)\frac{\alpha}{2}} \right) (\bar{\eta}_6 - (1 + \kappa_1 - \kappa_1\lambda_m + \lambda_m\nu)\bar{\eta}_7) \sin(\lambda_m + 1)\theta \right. \\ \left. - \bar{\eta}_7 \sin(\lambda_m - 1)\theta \right\},$
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Table 1(Continued)

<p>S(II)-S(II) $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	$\psi_{\theta 0}^{(m)} = A_4 r^{\lambda_m} \left\{ -\frac{1}{\lambda_m(\nu-1)} \left(\frac{\sin(\lambda_m-1)\alpha/2}{\sin(\lambda_m+1)\alpha/2} \right) (\bar{\eta}_6 - (1+\kappa_1 - \kappa_1\lambda_m + \lambda_m\nu)\bar{\eta}_7) \cos(\lambda_m+1)\theta \right.$ $\left. + (-\kappa_1\bar{\eta}_7 - \kappa_3\bar{\eta}_3) \cos(\lambda_m-1)\theta \right\},$ <p>where</p> $\bar{\eta}_6 = -\frac{\kappa_1(-1+\lambda_m)\bar{E}_1}{\bar{E}_2 - C_1\bar{E}_4} + \frac{(1+\lambda_m)\bar{E}_1}{\bar{E}_2 - C_1\bar{E}_4} + \frac{C_1(1+\lambda_m)^2\bar{E}_4\bar{\eta}_3}{(-\bar{E}_2 + C_1\bar{E}_4)}$ $+ \frac{\bar{\eta}_3}{(\bar{E}_2 - C_1\bar{E}_4)} (-1+\lambda_m)[\kappa_2\bar{E}_1 + \kappa_3\bar{E}_2 + C_1(-1-\kappa_3 + \lambda_m)\bar{E}_4],$ $\bar{\eta}_7 = -\left\{ 2\kappa_1(-1+\lambda_m)\bar{E}_1[\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)] \cos[(1+\lambda_m)\alpha/2] \sin[(-1+\lambda_m)\alpha/2] \right.$ $+ (1+\lambda_m)\{C_1(-1+\nu)\bar{E}_3(-\bar{E}_2 + C_1\bar{E}_4)$ $+ \bar{E}_1[(1+\nu)\bar{E}_2 + C_1(-3+\nu)\bar{E}_4 + 2C_1\bar{E}_6]\} \cos[(1+\lambda_m)\alpha/2] \sin[(-1+\lambda_m)\alpha/2]$ $+ (-1+\lambda_m)(-1+\nu)(\bar{E}_1 - C_1\bar{E}_3)(-\bar{E}_2 + C_1\bar{E}_4) \cos[(-1+\lambda_m)\alpha/2] \sin[(1+\lambda_m)\alpha/2]$ $- \bar{\eta}_3\{2(-1+\lambda_m)[\kappa_2\bar{E}_1 + \kappa_3\bar{E}_2 + C_1(-1-\kappa_3 + \lambda_m)\bar{E}_4]$ $[\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)] \cos[(1+\lambda_m)\alpha/2] \sin[(-1+\lambda_m)\alpha/2] - 2C_1(1+\lambda_m)$ $[(-1+\nu)(\bar{E}_2 - C_1\bar{E}_4)(-\bar{E}_4 + C_1\bar{E}_6) + (1+\lambda_m)\bar{E}_4(\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6))]$ $\cos[(1+\lambda_m)\alpha/2] \sin[(-1+\lambda_m)\alpha/2] + (1-\lambda_m)(-1+\nu)(\bar{E}_2 - C_1\bar{E}_4)$ $- \kappa_3\bar{E}_2 + C_1[2(1+\kappa_3)\bar{E}_4 - C_1(2+\kappa_3)\bar{E}_6]\} \cos[(-1+\lambda_m)\alpha/2] \sin[(1+\lambda_m)\alpha/2] \}$ $/ \left\{ (\bar{E}_2 - C_1\bar{E}_4)[\bar{E}_2 + C_1(-2\bar{E}_4 + C_1\bar{E}_6)] \{2(1+\kappa_1 - \kappa_1\lambda_m + \lambda_m\nu) \right.$ $\cos[(1+\lambda_m)\alpha/2] \sin[(-1+\lambda_m)\alpha/2] -$ $(1+\kappa_1)(-1+\lambda_m)(-1+\nu) \cos[(-1+\lambda_m)\alpha/2] \sin[(1+\lambda_m)\alpha/2] \}$
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Table 1(Continued)

<p style="text-align: center;">F- F $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>(1) Symmetrical case</p> <p>When $\lambda_m(-1+\nu)\sin\alpha + (3+\nu)\sin\lambda_m\alpha = 0$ or $\lambda_m\sin\alpha + \sin\lambda_m\alpha = 0$,</p> $u_0^{(m)} = D_3 r^{\lambda_m} \left\{ \left(\frac{\bar{\eta}_{10}\bar{\eta}_8}{\sin(\lambda_m+1)\alpha/2} (\sin\alpha + \sin\lambda_m\alpha) \right) \cos(\lambda_m+1)\theta - \bar{\eta}_8 \cos(\lambda_m-1)\theta \right\},$ $v_0^{(m)} = D_3 r^{\lambda_m} \left\{ - \left(\frac{\bar{\eta}_{10}\bar{\eta}_8}{\sin(\lambda_m+1)\alpha/2} (\sin\alpha + \sin\lambda_m\alpha) \right) \sin(\lambda_m+1)\theta \right.$ $\left. + \left(-\kappa_1\bar{\eta}_8 - \kappa_2 \frac{\bar{\eta}_9}{C_1(\lambda_m+1)} \right) \sin(\lambda_m-1)\theta \right\},$ $w_0^{(m)} = D_3 r^{\lambda_m+1} \left\{ \bar{\eta}_9 \left[\frac{\bar{\eta}_{10}}{C_1(\lambda_m+1)} \left(\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \cos(\lambda_m+1)\theta \right. \right.$ $\left. \left. - \frac{1}{C_1(\lambda_m+1)} \cos(\lambda_m-1)\theta \right] \right\},$ $\psi_{r0}^{(m)} = D_3 r^{\lambda_m} \left\{ -\bar{\eta}_{10} \left(\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \cos(\lambda_m-1)\theta + \cos(\lambda_m+1)\theta \right\},$ $\psi_{\theta0}^{(m)} = D_3 r^{\lambda_m} \left\{ \bar{\eta}_{10} \left(\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \sin(\lambda_m+1)\theta + \right.$ $\left. + \left(-\kappa_1 - \kappa_3 \frac{\bar{\eta}_9}{C_1(\lambda_m+1)} \right) \sin(\lambda_m-1)\theta \right\},$ <p>where</p> $\bar{\eta}_8 = \frac{\bar{E}_2\bar{E}_3 - C_1\bar{E}_3\bar{E}_4 + \bar{E}_1(-\bar{E}_4 + C_1\bar{E}_6))}{\bar{E}_1\bar{E}_3 - C_1\bar{E}_3^2 + \bar{E}_0(-\bar{E}_4 + C_1\bar{E}_6))},$ $\bar{\eta}_9 = \frac{\bar{E}_1^2 - 2C_1\bar{E}_1\bar{E}_3 + C_1^2\bar{E}_3^2 - \bar{E}_0(\bar{E}_2 - 2C_1\bar{E}_4 + C_1^2\bar{E}_6))}{(-\bar{E}_1\bar{E}_3 + C_1\bar{E}_3^2 + \bar{E}_0(\bar{E}_4 - C_1\bar{E}_6))},$ $\bar{\eta}_{10} = \frac{(3+\lambda_m(\nu-1)+\nu)}{(\lambda_m+1)(\nu-1)}.$
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Table 1(Continued)

<p style="text-align: center;">F- F $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>(2) Anti-symmetrical case When $-\lambda_m(-1+\nu)\sin\alpha+(3+\nu)\sin\lambda_m\alpha=0$ or $-\lambda_m\sin\alpha+\sin\lambda_m\alpha=0$,</p> $u_0^{(m)} = D_4 r^{\lambda_m} \left\{ - \left(\frac{\bar{\eta}_{10}\bar{\eta}_8}{\sin(\lambda_m+1)\alpha/2} (\sin\alpha - \sin\lambda_m\alpha) \right) \sin(\lambda_m+1)\theta - \bar{\eta}_8 \sin(\lambda_m-1)\theta \right\},$ $v_0^{(m)} = D_4 r^{\lambda_m} \left\{ - \left(\frac{\bar{\eta}_{10}\bar{\eta}_8}{\sin(\lambda_m+1)\alpha/2} (\sin\alpha - \sin\lambda_m\alpha) \right) \cos(\lambda_m+1)\theta \right. \\ \left. + \left(-\kappa_1\bar{\eta}_8 - \kappa_2 \frac{\bar{\eta}_9}{C_1(\lambda_m+1)} \right) \cos(\lambda_m-1)\theta \right\},$ $w_0^{(m)} = D_4 r^{\lambda_m+1} \left\{ \bar{\eta}_9 \left[\frac{\bar{\eta}_{10}}{C_1(\lambda_m+1)} \left(\frac{\sin(\lambda_m-1)\alpha/2}{\sin(\lambda_m+1)\alpha/2} \right) \sin(\lambda_m+1)\theta \right. \right. \\ \left. \left. - \frac{1}{C_1(\lambda_m+1)} \sin(\lambda_m-1)\theta \right] \right\},$ $\psi_{r0}^{(m)} = D_4 r^{\lambda_m} \left\{ -\bar{\eta}_{10} \left(\frac{\sin(\lambda_m-1)\alpha/2}{\sin(\lambda_m+1)\alpha/2} \right) \sin(\lambda_m+1)\theta + \sin(\lambda_m-1)\theta \right\},$ $\psi_{\theta0}^{(m)} = D_4 r^{\lambda_m} \left\{ -\bar{\eta}_{10} \left(\frac{\sin(\lambda_m-1)\alpha/2}{\sin(\lambda_m+1)\alpha/2} \right) \cos(\lambda_m+1)\theta \right. \\ \left. + \left(\kappa_1 - \kappa_3 \frac{\bar{\eta}_9}{C_1(\lambda+1)} \right) \cos(\lambda_m-1)\theta \right\},$
<p style="text-align: center;">C- C $(-\frac{\alpha}{2} \leq \theta \leq \frac{\alpha}{2})$</p>	<p>(1) Symmetrical case When $\lambda_m(1+\nu)\sin\alpha+(-3+\nu)\sin\lambda_m\alpha=0$ or $\lambda_m\sin\alpha+\sin\lambda_m\alpha=0$,</p> $u_0^{(m)} = D_3 r^{\lambda_m} \left\{ \frac{\kappa_2}{\kappa_3} \left[\left(-\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \cos(\lambda_m+1)\theta + \cos(\lambda_m-1)\theta \right] \right\},$ $v_0^{(m)} = D_3 r^{\lambda_m} \left\{ \frac{\kappa_2}{\kappa_3} \left[\left(\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \sin(\lambda_m+1)\theta + (-\kappa_1 + \kappa_2\bar{\eta}_2) \sin(\lambda_m-1)\theta \right] \right\},$ $w_0^{(m)} = D_3 r^{\lambda_m+1} \left\{ \frac{\kappa_2}{\kappa_3} \bar{\eta}_2 \left[\left(-\frac{\cos(\lambda_m-1)\alpha/2}{\cos(\lambda_m+1)\alpha/2} \right) \cos(\lambda_m+1)\theta + \cos(\lambda_m-1)\theta \right] \right\},$

Table 1(Continued)

	$\psi_{r_0}^{(m)} = D_3 r^{\lambda_m} \left\{ \left(-\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m - 1)\theta + \cos(\lambda_m + 1)\theta \right\},$ $\psi_{\theta_0}^{(m)} = D_3 r^{\lambda_m} \left\{ \left(\frac{\cos(\lambda_m - 1)\alpha/2}{\cos(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + (-\kappa_1 + \kappa_2 \bar{\eta}_2) \sin(\lambda_m - 1)\theta \right\},$ <p>(2) Anti-symmetrical case</p> <p>When $\lambda_m(1+\nu) \sin \alpha - (-3+\nu) \sin \lambda_m \alpha = 0$ or $-\lambda_m \sin \alpha + \sin \lambda_m \alpha = 0$,</p> $u_0^{(m)} = D_4 r^{\lambda_m} \left\{ \frac{\kappa_2}{\kappa_3} \left[\left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right] \right\},$ $v_0^{(m)} = D_4 r^{\lambda_m} \left\{ \frac{\kappa_2}{\kappa_3} \left[\left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + (\kappa_1 - \kappa_2 \bar{\eta}_3) \cos(\lambda_m - 1)\theta \right] \right\},$ $w_0^{(m)} = D_4 r^{\lambda_m+1} \left\{ \frac{\kappa_2}{\kappa_3} \bar{\eta}_3 \left[\left(-\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right] \right\},$ $\psi_{r_0}^{(m)} = D_4 r^{\lambda_m} \left\{ -\left(\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \sin(\lambda_m + 1)\theta + \sin(\lambda_m - 1)\theta \right\},$ $\psi_{\theta_0}^{(m)} = D_4 r^{\lambda_m} \left\{ \left(\frac{\sin(\lambda_m - 1)\alpha/2}{\sin(\lambda_m + 1)\alpha/2} \right) \cos(\lambda_m + 1)\theta + (\kappa_1 - \kappa_2 \bar{\eta}_3) \cos(\lambda_m - 1)\theta \right\}$
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Table 2 Characteristic equations

Boundary conditions	Characteristic equations
S(I)- S(I)	<p>Symmetry:</p> $\cos \alpha + \cos \lambda_m \alpha = 0^* ; \quad \bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0 ;$ <p>where $\bar{\gamma}_1 = \frac{\bar{\kappa}_1}{\bar{\kappa}_2}$;</p> $\bar{\kappa}_1 = [-C_1 \bar{E}_4 (-\kappa_2 \bar{E}_3 + 2(1 + \kappa_1) \bar{E}_4) + \kappa_2 \bar{E}_1 (\bar{E}_4 - C_1 \bar{E}_6) + \bar{E}_2 (-\kappa_2 \bar{E}_3 + 2C_1 (1 + \kappa_1) \bar{E}_6)] ;$ $\bar{\kappa}_2 = [C_1 \bar{E}_4 (-\kappa_2 \bar{E}_3 + 2(-1 + \kappa_1) \lambda \bar{E}_4) - \kappa_2 \bar{E}_1 (\bar{E}_4 - C_1 \bar{E}_6) - \bar{E}_2 (-\kappa_2 \bar{E}_3 + 2C_1 (-1 + \kappa_1) \lambda \bar{E}_6)] ;$ <p>Antisymmetry:</p> $\cos \alpha - \cos \lambda \alpha = 0^* ; \quad \sin \lambda_m \alpha - \bar{\gamma}_1 \lambda \sin \alpha = 0$
S(II)-S(II)	<p>Symmetry:</p> $\cos \alpha + \cos \lambda_m \alpha = 0^* ; \quad \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0^* ;$ $\bar{\gamma}_1 \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0 ;$ <p>Antisymmetry:</p> $\cos \alpha - \cos \lambda_m \alpha = 0^* ; \quad \lambda_m \sin \alpha - \sin \lambda_m \alpha = 0^* ;$ $\sin \lambda_m \alpha - \bar{\gamma}_1 \lambda_m \sin \alpha = 0$
S(I)- S(II)	$\cos 2\alpha - \cos 2\lambda_m \alpha = 0^* ; \quad \lambda_m \sin 2\alpha - \sin 2\lambda_m \alpha = 0^* ;$ $\bar{\gamma}_2 \lambda_m^2 \sin^2 \alpha + \sin^2 \lambda_m \alpha = 0 ;$ <p>where $\bar{\gamma}_2 = \frac{\bar{\kappa}_3}{\bar{\kappa}_4}$,</p> $\bar{\kappa}_3 = [-2C_1 \bar{E}_1 \bar{E}_3 \bar{E}_4 ((3 + \nu) \bar{E}_4 - 2C_1 \bar{E}_6) + \bar{E}_2^2 (2\bar{E}_3^2 - (1 + \nu) \bar{E}_0 \bar{E}_6) + \bar{E}_1^2 (-(\nu - 1) \bar{E}_4^2 - 4C_1 \bar{E}_4 \bar{E}_6 + 2C_1^2 \bar{E}_6^2) + C_1 \bar{E}_4^2 (C_1 (1 - \nu) \bar{E}_3^2 - (1 + \nu) \bar{E}_0 (2\bar{E}_4 - C_1 \bar{E}_6)) + \bar{E}_2 ((1 + \nu) \bar{E}_1^2 \bar{E}_6 - 2\bar{E}_1 \bar{E}_3 (2\bar{E}_4 + C_1 (-1 + \nu) \bar{E}_6) + C_1 \bar{E}_3^2 (-4\bar{E}_4 + C_1 (1 + \nu) \bar{E}_6) + (1 + \nu) \bar{E}_0 (\bar{E}_4^2 + 2C_1 \bar{E}_4 \bar{E}_6 - C_1^2 \bar{E}_6^2))]^2 ,$ $\bar{\kappa}_4 = [-2C_1 \bar{E}_1 \bar{E}_3 \bar{E}_4 ((-5 + \nu) \bar{E}_4 + 2C_1 \bar{E}_6) + \bar{E}_2^2 (2\bar{E}_3^2 + (-3 + \nu) \bar{E}_0 \bar{E}_6) + \bar{E}_1^2 ((\nu - 1) \bar{E}_4^2 - 4C_1 \bar{E}_4 \bar{E}_6 + 2C_1^2 \bar{E}_6^2) + C_1 \bar{E}_4^2 (C_1 (-1 + \nu) \bar{E}_3^2 + (-3 + \nu) \bar{E}_0 (2\bar{E}_4 - C_1 \bar{E}_6)) - \bar{E}_2 ((-3 + \nu) \bar{E}_1^2 \bar{E}_6 + C_1 \bar{E}_3^2 (4\bar{E}_4 + C_1 (-3 + \nu) \bar{E}_6) + 2\bar{E}_1 \bar{E}_3 (2\bar{E}_4 - C_1 (-1 + \nu) \bar{E}_6) + (-3 + \nu) \bar{E}_0 (\bar{E}_4^2 + 2C_1 \bar{E}_4 \bar{E}_6 - C_1^2 \bar{E}_6^2))]^2 .$
F- F	<p>Symmetry:</p> $\lambda_m (-1 + \nu) \sin \alpha + (3 + \nu) \sin \lambda_m \alpha = 0^* ; \quad \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0^{*,\#} ;$ <p>Antisymmetry:</p> $-\lambda_m (-1 + \nu) \sin \alpha + (3 + \nu) \sin \lambda_m \alpha = 0^* ; \quad -\lambda_m \sin \alpha + \sin \lambda_m \alpha = 0^{*,\#} .$
C- C	<p>Symmetry:</p> $\lambda_m (1 + \nu) \sin \alpha + (-3 + \nu) \sin \lambda_m \alpha = 0^{*,\#} ; \quad \lambda_m \sin \alpha + \sin \lambda_m \alpha = 0^* ;$ <p>Antisymmetry:</p> $\lambda_m (1 + \nu) \sin \alpha - (-3 + \nu) \sin \lambda_m \alpha = 0^{*,\#} ; \quad -\lambda_m \sin \alpha + \sin \lambda_m \alpha = 0^* .$

C- F	$4 - \lambda_m^2 (1 + \nu)^2 \sin^2 \alpha + (-3 + \nu)(1 + \nu) \sin^2 \lambda_m \alpha = 0^{*,\#};$ $4 - \lambda_m^2 (1 - \nu)^2 \sin^2 \alpha + (3 + \nu)(-1 + \nu) \sin^2 \lambda_m \alpha = 0^*.$
S(II)- C	$4 - \lambda_m^2 (1 + \nu)^2 \sin^2 \alpha + (-3 + \nu)(1 + \nu) \sin^2 \lambda_m \alpha = 0^*;$ $\{[2\bar{E}_1(\bar{E}_4 - C_1\bar{E}_6) \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]$ $+ \bar{E}_4[4\bar{E}_4(\lambda_m^2 (1 + \nu)^2 \sin^2 \alpha - (-3 + \nu)^2 \sin^2 \lambda_m \alpha)(\lambda_m \sin 2\alpha - \sin 2\lambda_m \alpha) + 2C_1\bar{E}_3 \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]$ $- \bar{E}_2[4\bar{E}_6(\lambda_m^2 (1 + \nu)^2 \sin^2 \alpha - (-3 + \nu)^2 \sin^2 \lambda_m \alpha)(\lambda_m \sin 2\alpha - \sin 2\lambda_m \alpha) + 2\bar{E}_3 \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]\} = 0$ <p>where $\hat{E}_1 = \bar{E}_1^2 - 2C_1\bar{E}_1\bar{E}_3 + C_1^2\bar{E}_3^2 - \bar{E}_0(\bar{E}_2 - 2C_1\bar{E}_4 + C_1^2\bar{E}_6),$ $\hat{E}_2 = -\bar{E}_2\bar{E}_3 + C_1\bar{E}_3\bar{E}_4 + \bar{E}_1(\bar{E}_4 - C_1\bar{E}_6),$ $\hat{f}_1 = \lambda_m^3 (1 + \nu) \sin 4\alpha - 2\lambda_m \sin 2\alpha (\lambda_m^2 (1 + \nu) + 2(-5 + 3\nu) \sin^2 \lambda_m \alpha)$ $+ 4(\lambda_m^2 (-1 + 3\nu) \sin^2 \alpha + (-3 + \nu) \sin^2 \lambda_m \alpha) \sin 2\lambda_m \alpha .$</p>
S(I)- C	$- \lambda_m (1 + \nu) \sin 2\alpha + (-3 + \nu) \sin 2\lambda_m \alpha = 0^*;$ $\{[2\bar{E}_1(\bar{E}_4 - C_1\bar{E}_6) \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]$ $+ \bar{E}_4[4\bar{E}_4(\lambda_m^2 (1 + \nu)^2 \sin^2 \alpha - (-3 + \nu)^2 \sin^2 \lambda_m \alpha)(\lambda_m \sin 2\alpha - \sin 2\lambda_m \alpha) + 2C_1\bar{E}_3 \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]$ $- \bar{E}_2[4\bar{E}_6(\lambda_m^2 (1 + \nu)^2 \sin^2 \alpha - (-3 + \nu)^2 \sin^2 \lambda_m \alpha)(\lambda_m \sin 2\alpha - \sin 2\lambda_m \alpha) + 2\bar{E}_3 \frac{\hat{E}_2}{\hat{E}_1} \hat{f}_1]\} = 0$

Note: * means that equation can be found in a homogeneous plate under bending.

means that equation can be found in a homogeneous plate under extension..

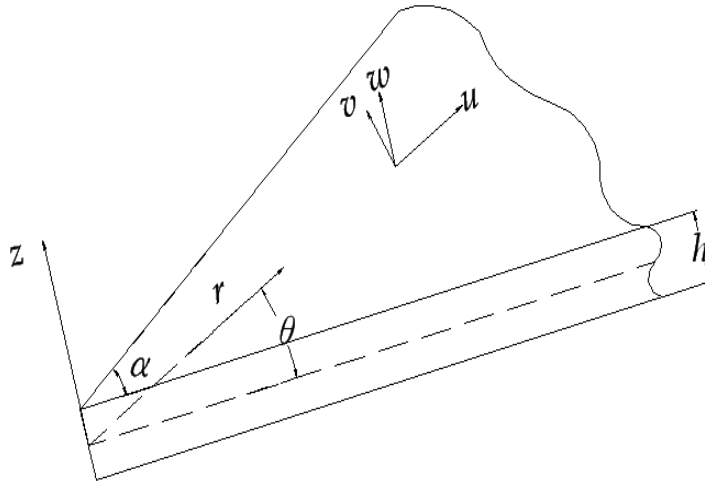


Fig. 1 Coordinate system and positive displacement components for a sector plate

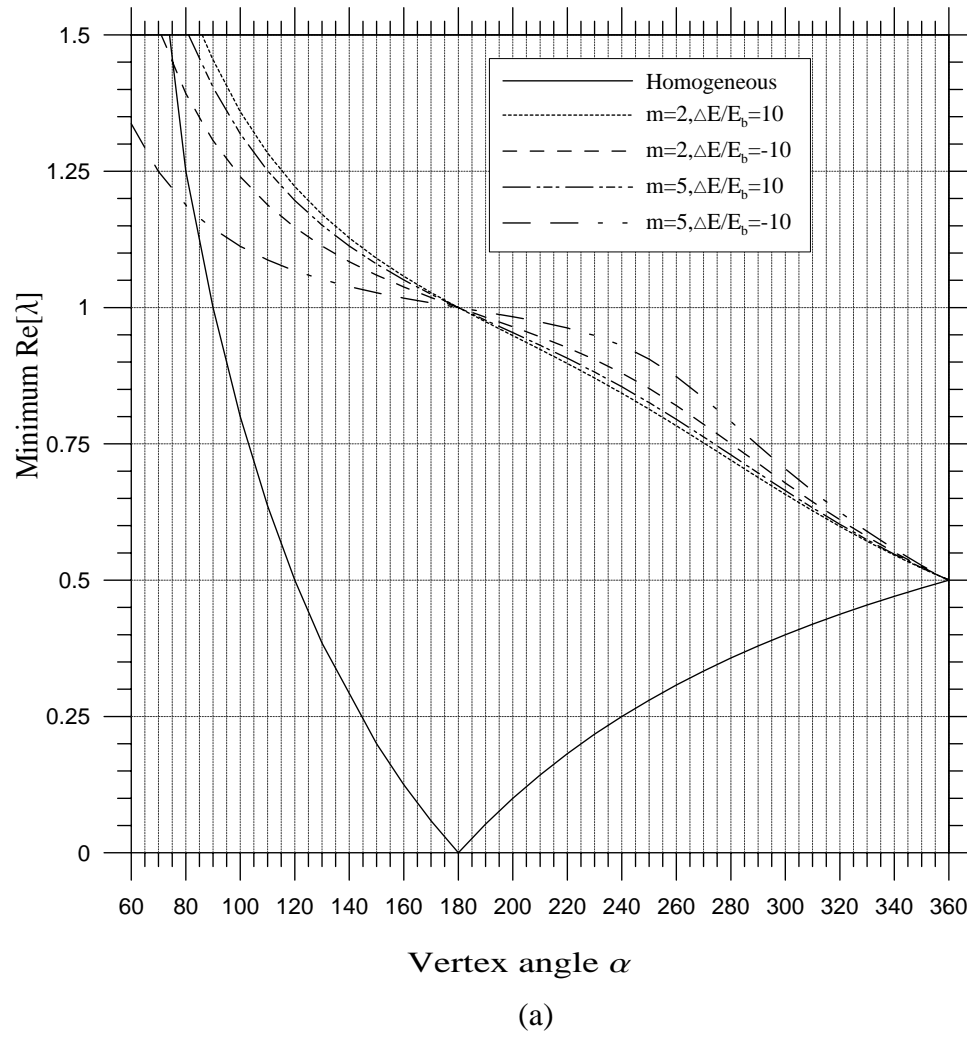
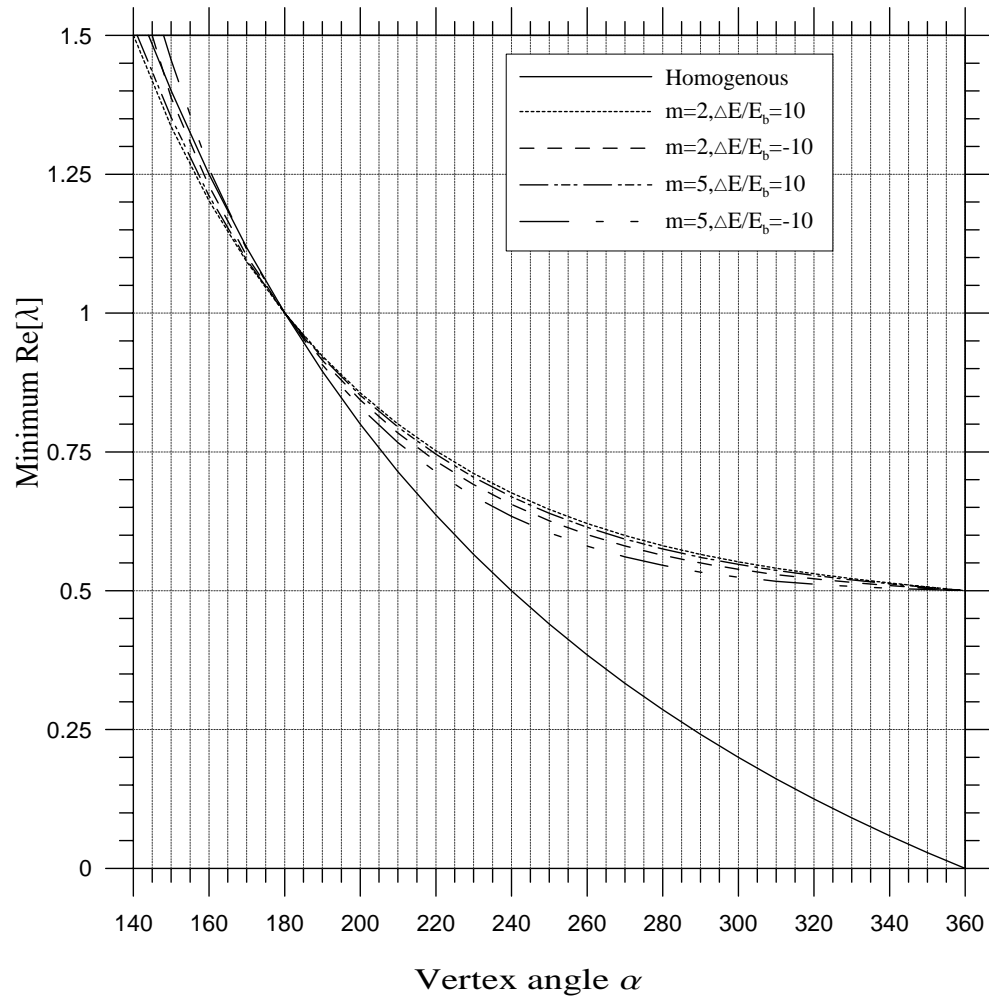


Fig.2 Minimum $\text{Re}[\lambda]$ of characteristic equations corresponding to S(I)-S(I) boundary condition: (a) symmetric solution, (b) anti-symmetric solution



(b)

Fig. 2 (Continued)

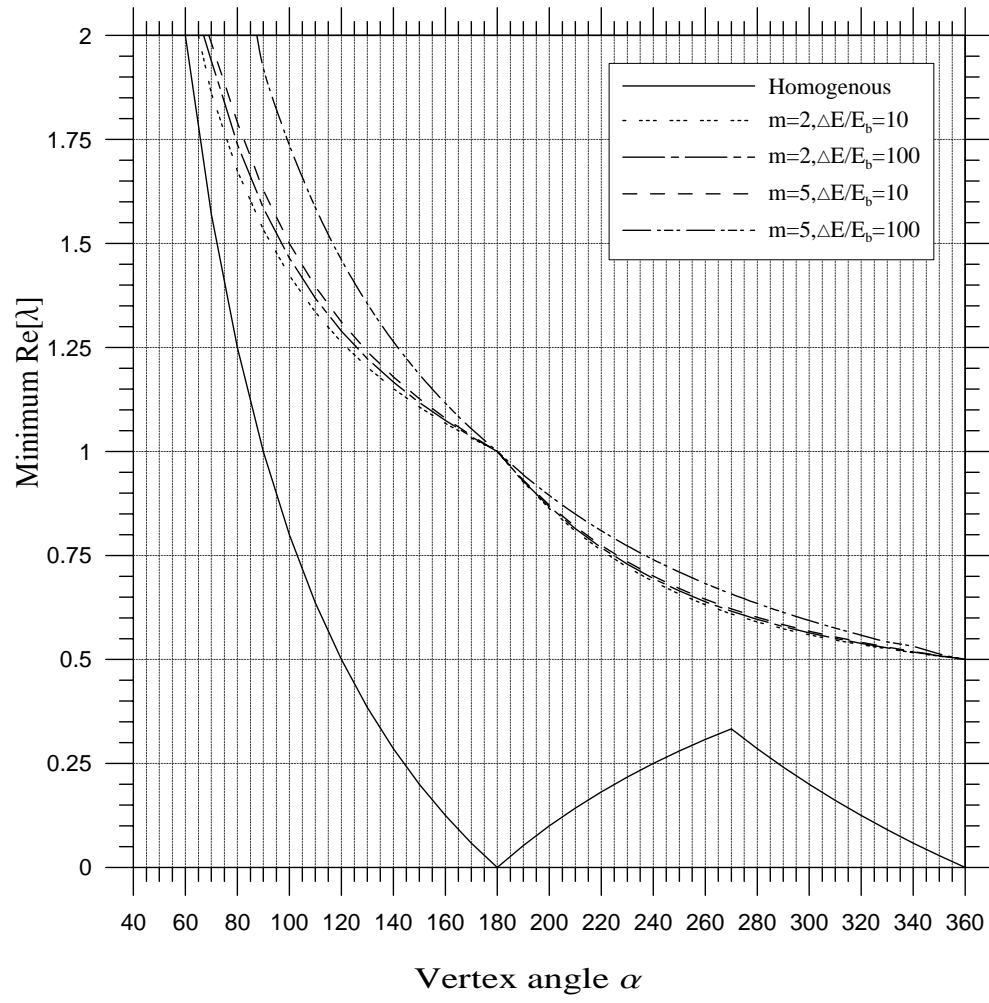


Fig.3 Minimum $\text{Re}[\lambda]$ of characteristic equations corresponding to S(I)-S(II) boundary condition.

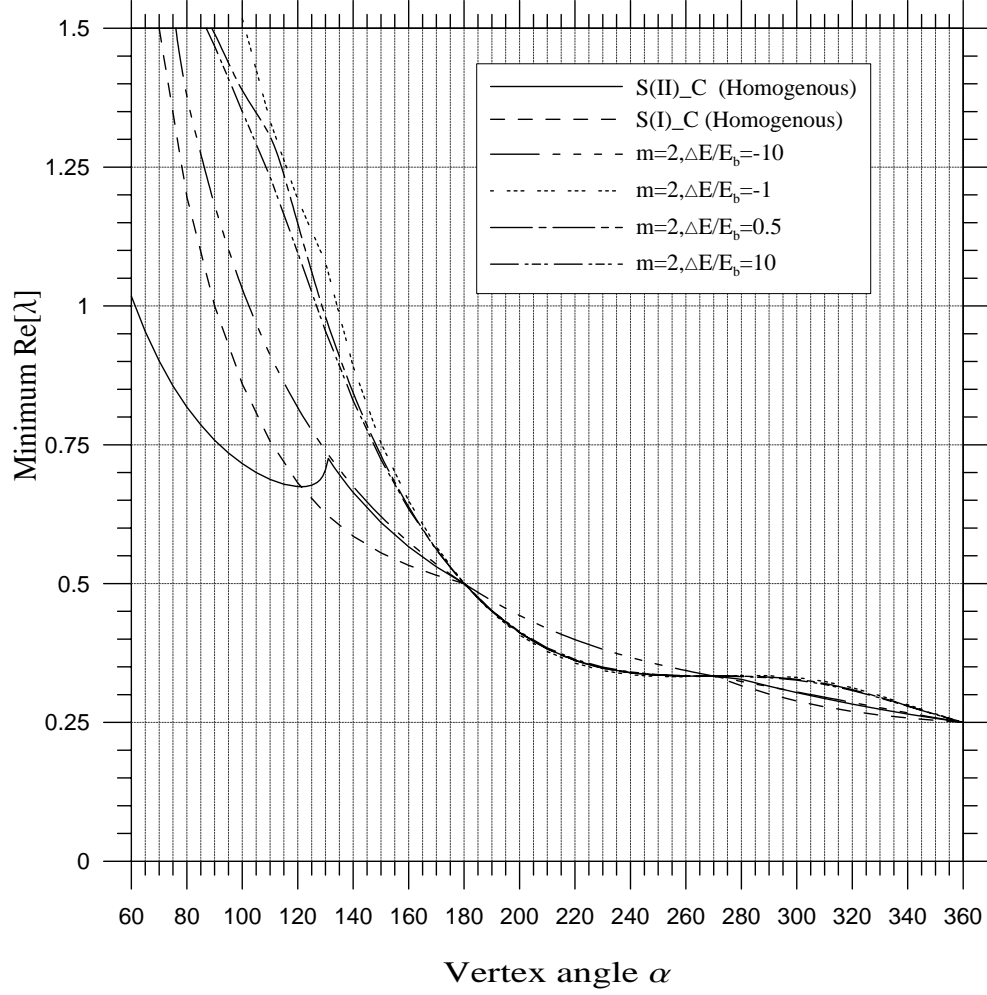


Fig. 4 Minimum $\text{Re}[\lambda]$ of characteristic equations corresponding to S(I)-C and S(II)-C boundary conditions.

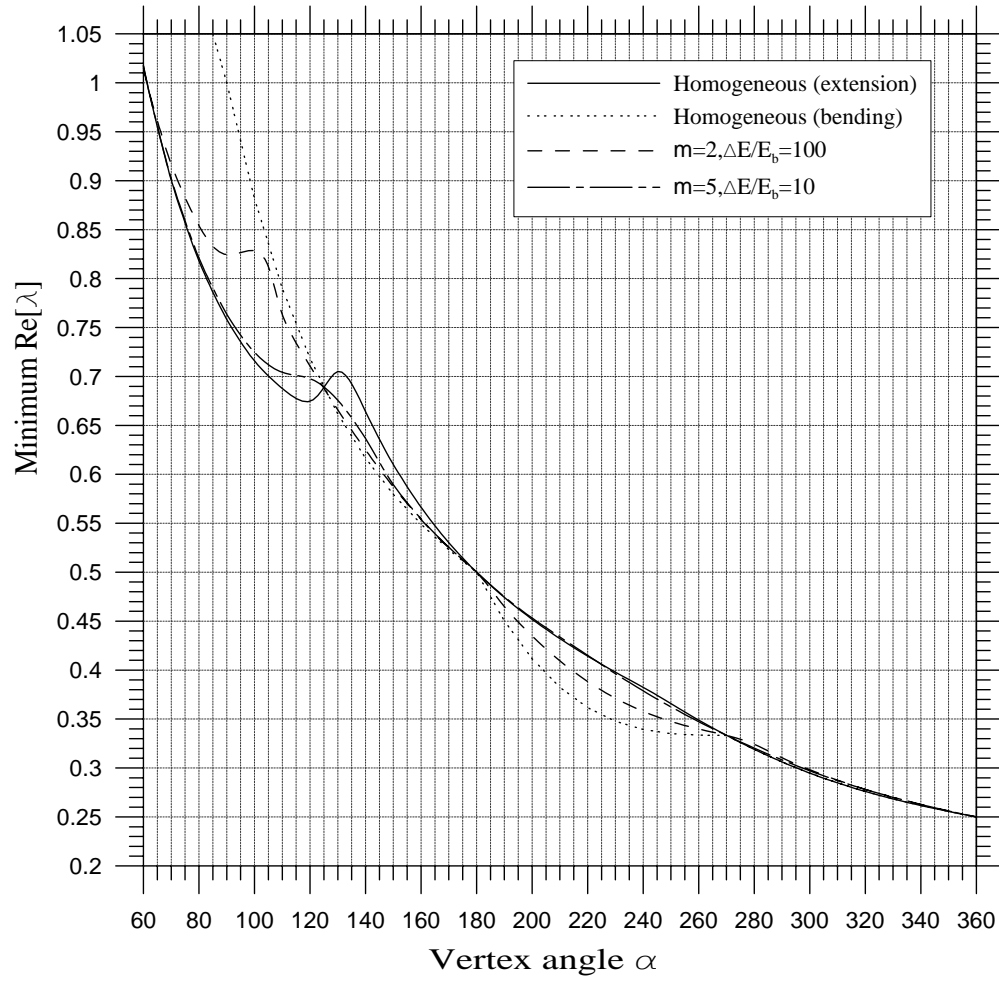


Fig. 5 Minimum $\text{Re}[\lambda]$ of characteristic equations corresponding to S(I)-F boundary condition.

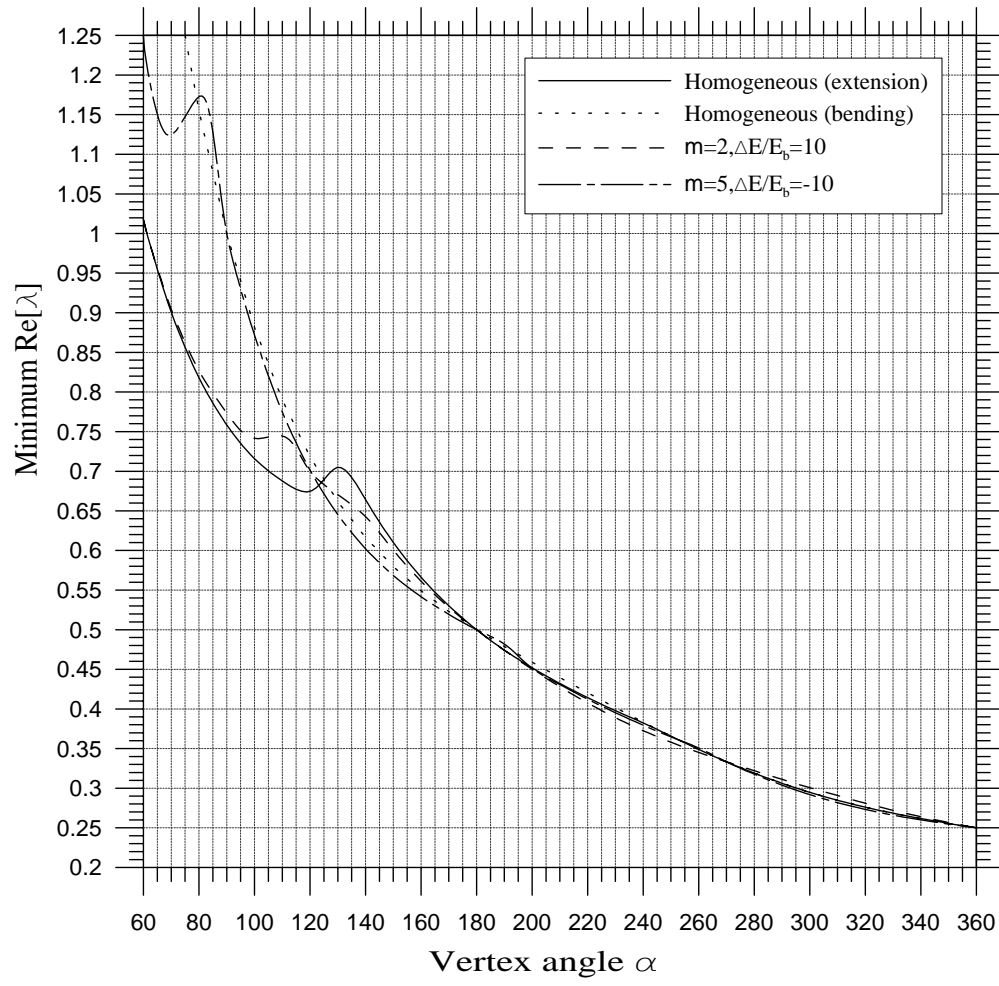


Fig. 6 Minimum $\text{Re}[\lambda]$ of characteristic equations corresponding to S(II)-F boundary condition.